

Heat Transfer and Working Temperature Field of a Photovoltaic Panel under Realistic Environmental Conditions

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Introduction

- ❑ A great portion of the solar radiation absorbed by a photovoltaic module (typically 85% of the incident radiation) **is not converted into electrical energy.**
- ❑ It is wasted by heat transfer with the surrounding medium, and also the **increase** of the module's temperature **reduce its efficiency.**
- ❑ The working temperature of photovoltaic modules **depends on different environmental factors:**

The ambient temperature.

The solar irradiation.

The relative humidity.

The direction and speed of the wind.

The construction materials.

The installation of the module.

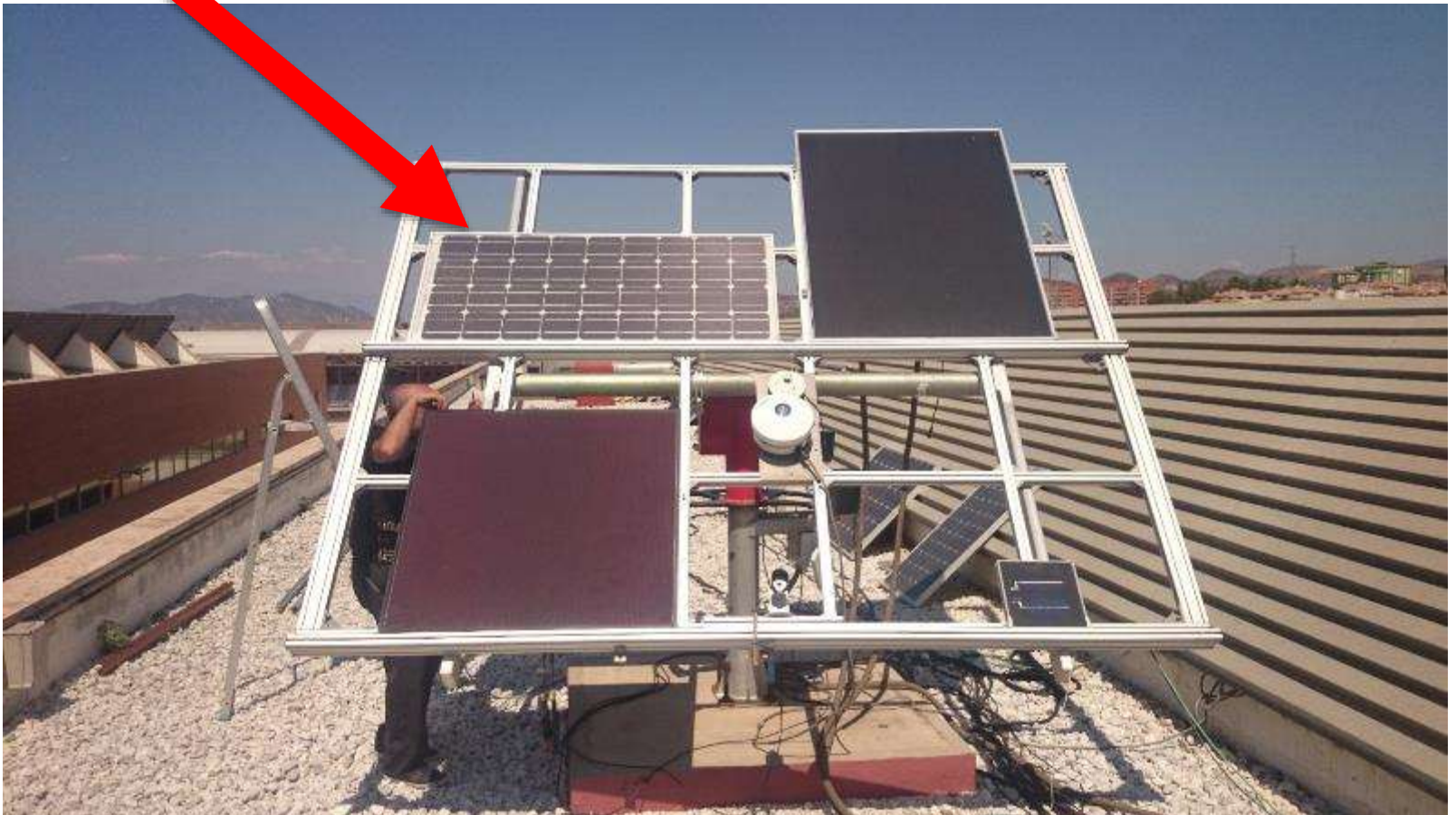
PRESENT WORK: We perform a **numerical study, using COMSOL Multiphysics**, of the convective heat transfer and transient temperature field of a photovoltaic module.

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Experimental system

- ❑ The PV panel is mounted in a sun tracker, so we assume normal incident direct radiation.
- ❑ Also, we have to take into account the diffuse radiation.

This one!



Experimental system

□ The module is installed with an angle 30° with the horizontal axis.

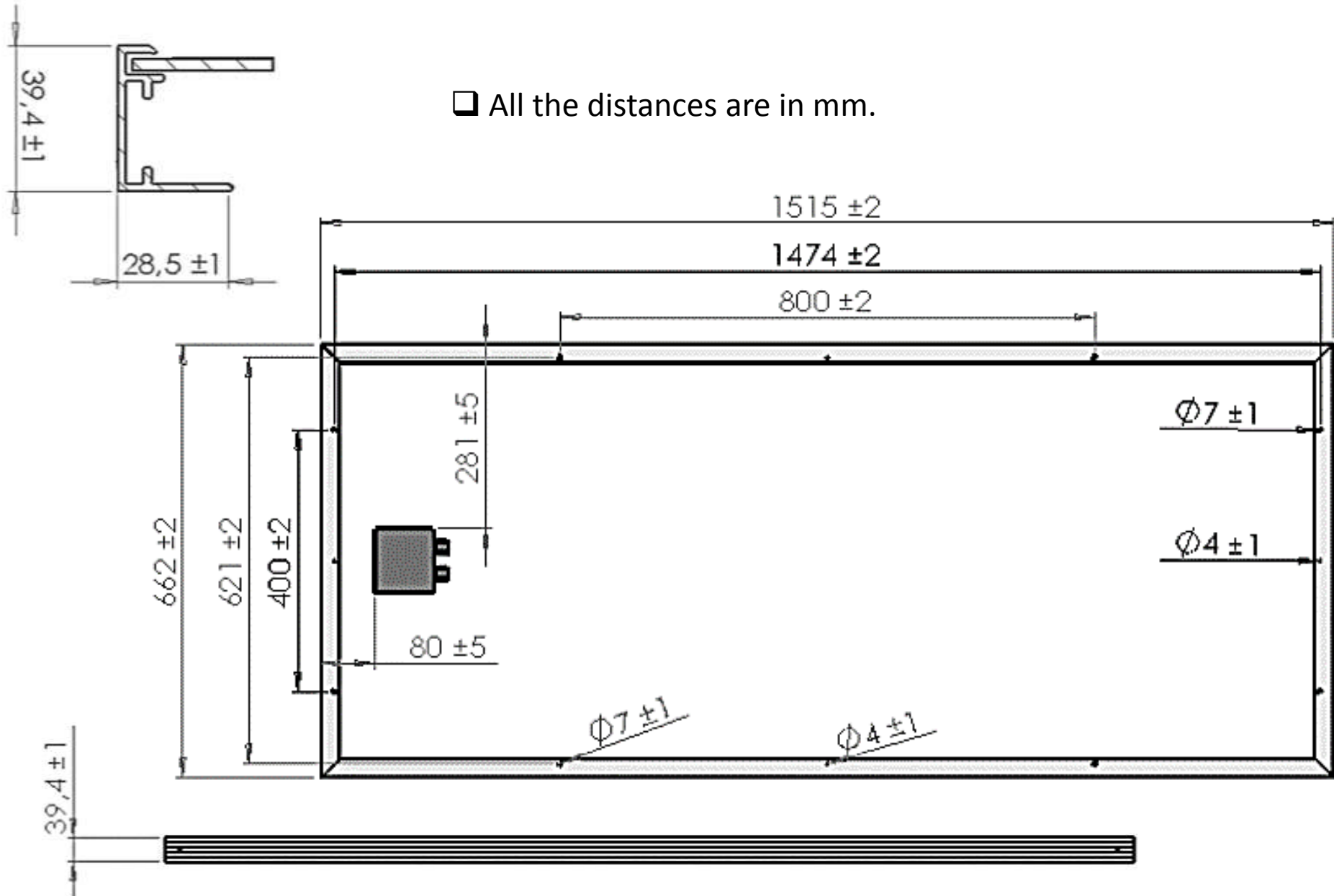


Solid parts

1. The glass of the cover. Thickness: 3 mm.
2. The Silicon cells. Thickness: 0.4 mm.
3. The EVA (ethylene vinyl acetate) film. Thickness: 0.8 mm.
4. The Tedlar back film. Thickness: 0.05 mm. White re
5. The aluminium frame of the PV panel.

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Experimental system

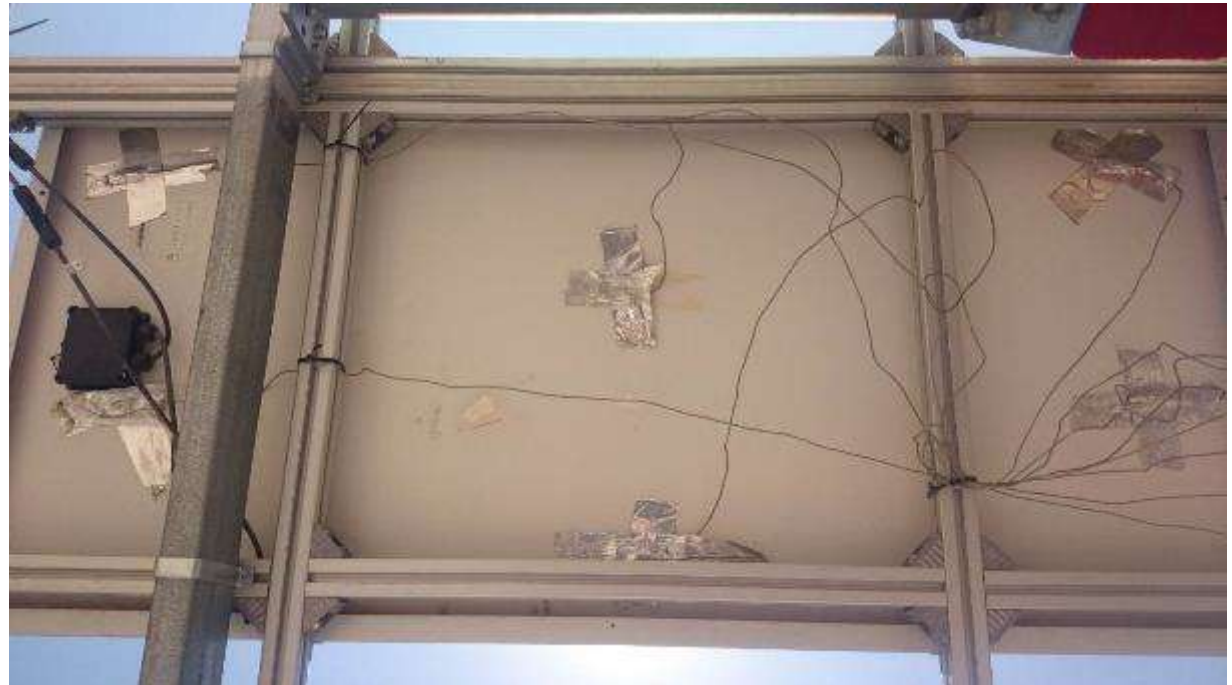
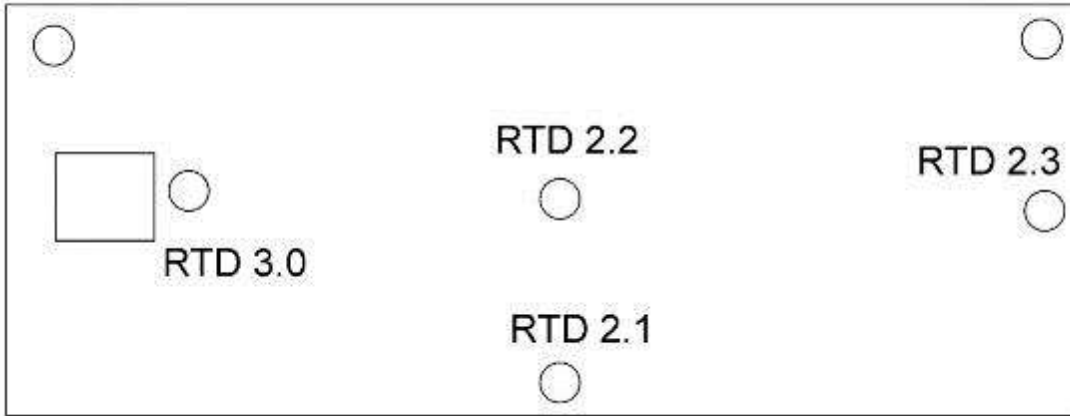


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Pt100 Temperature sensors

RTD 2.0

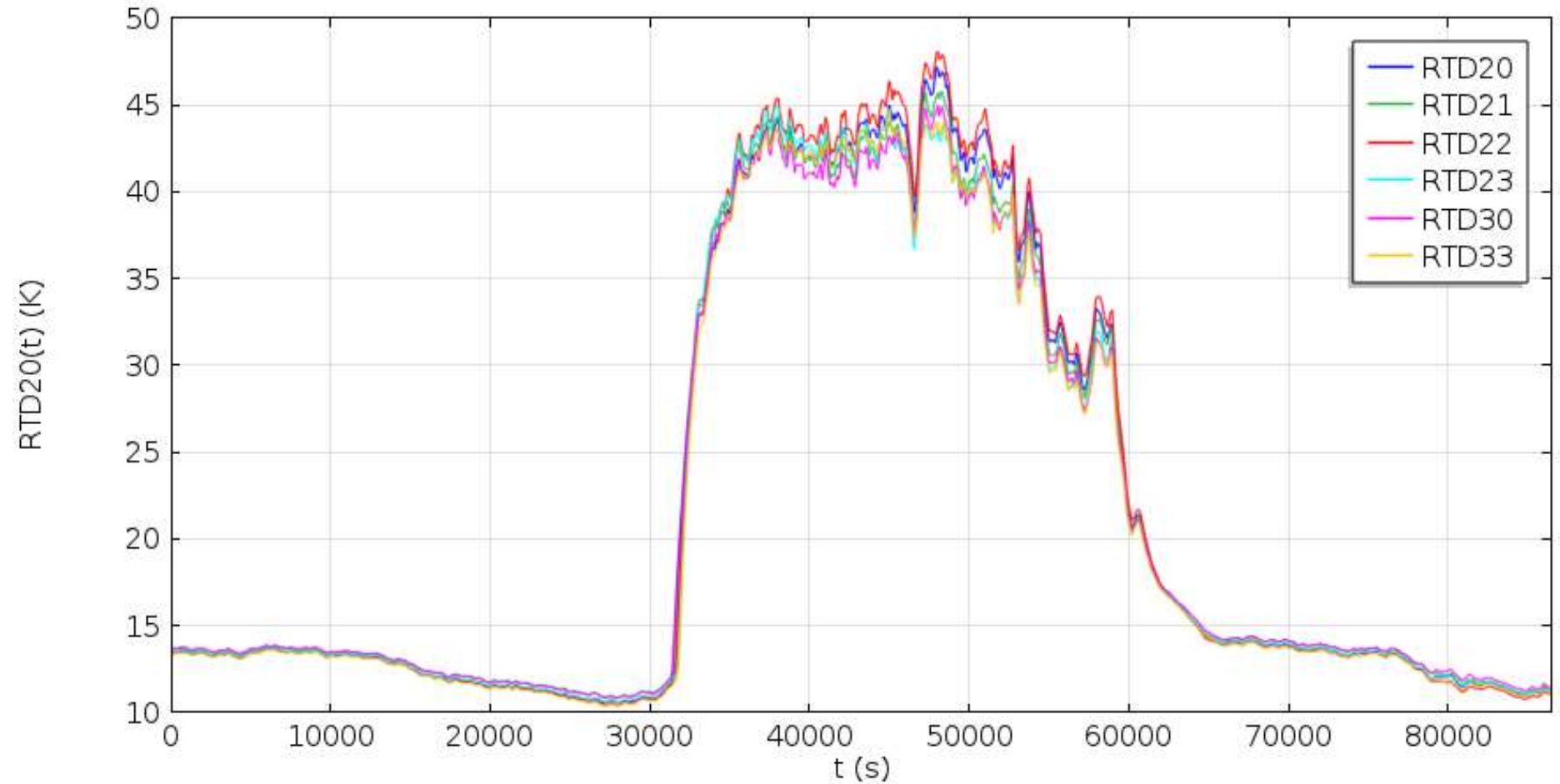
RTD 3.3



Pt100 Temperature sensors

RTD20(t) (K)

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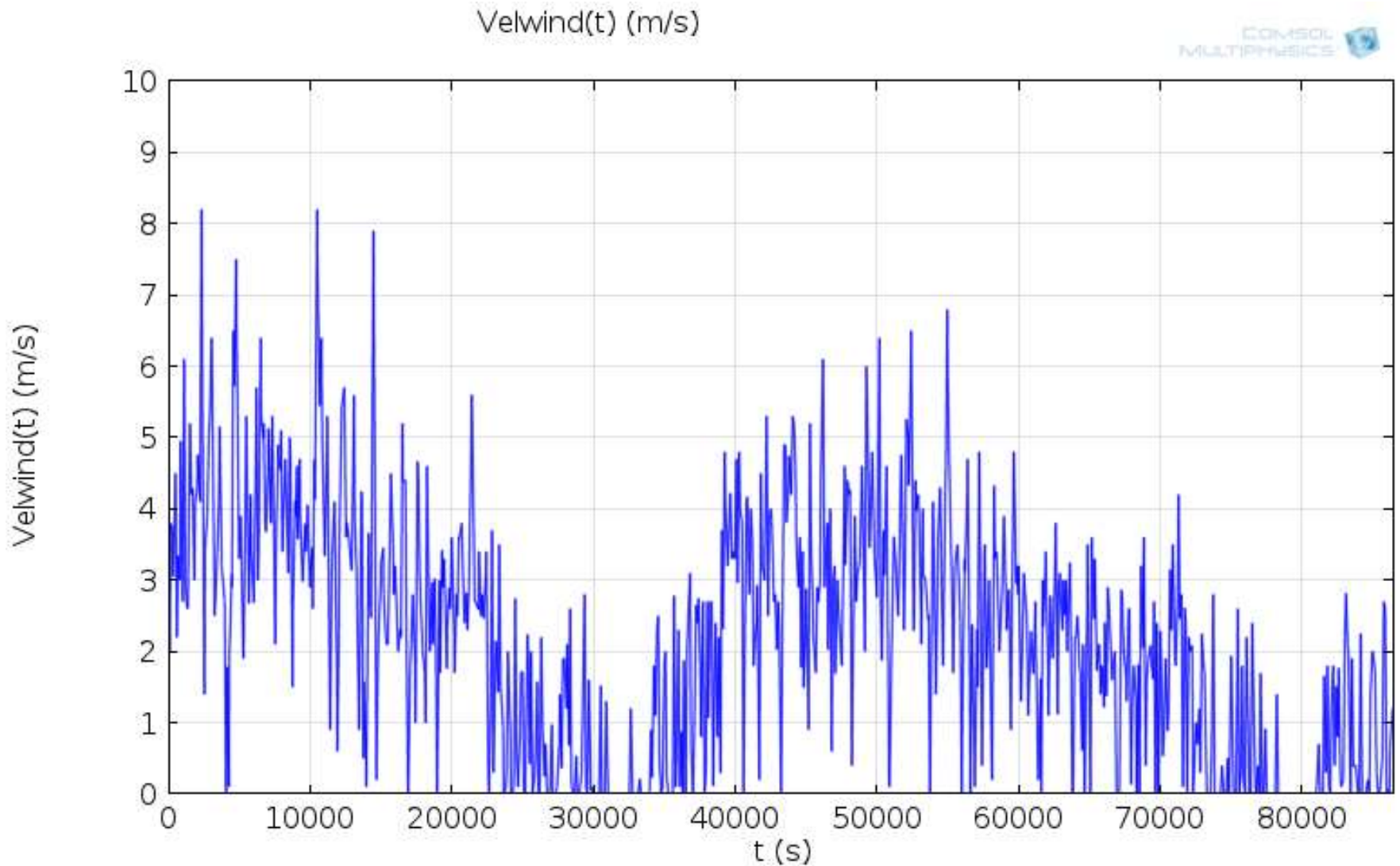


8 Irradiation and meteorological measurements



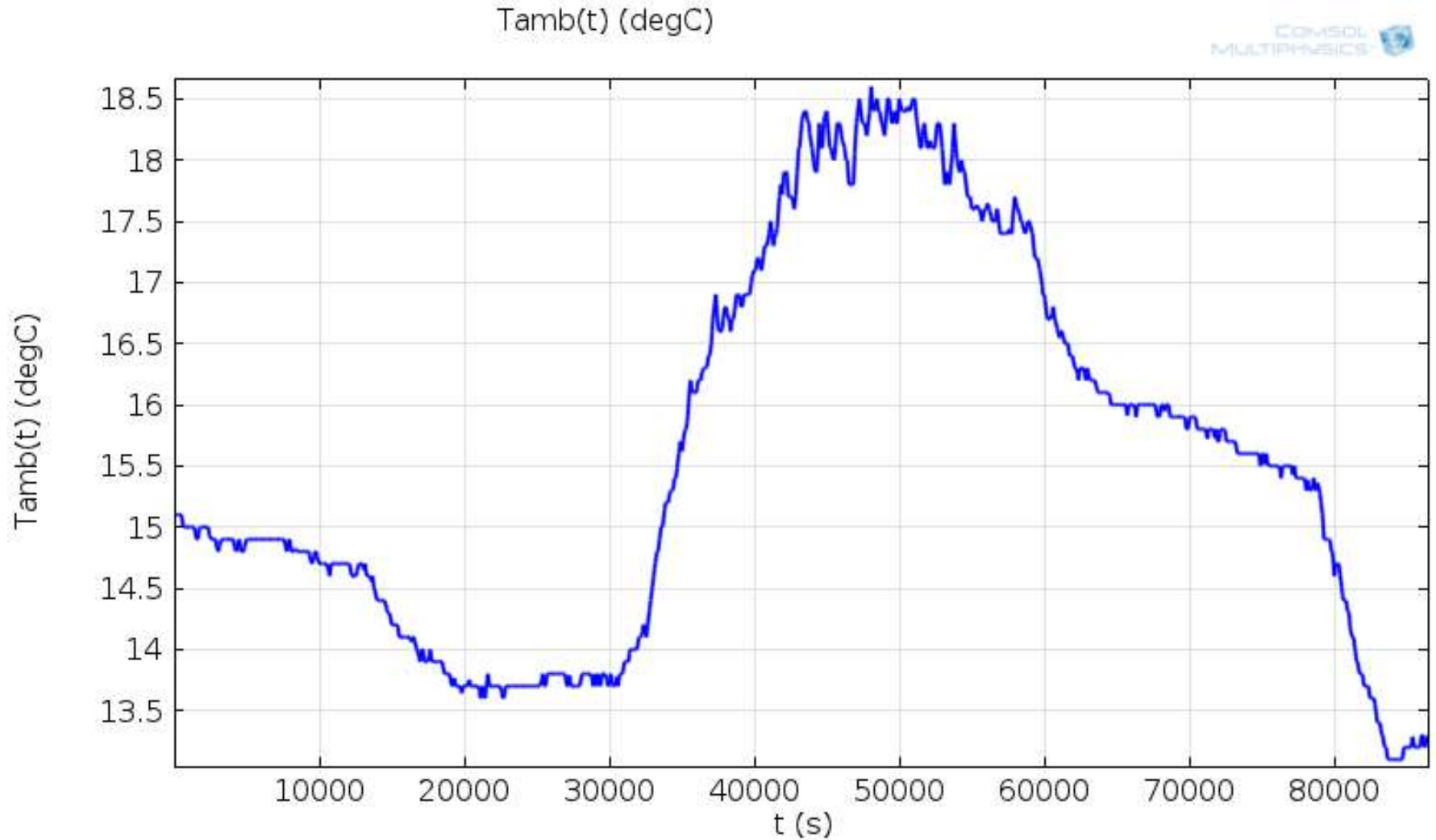
☐ Pyranometers, anemometer and wind vane.

9 Irradiation and meteorological measurements

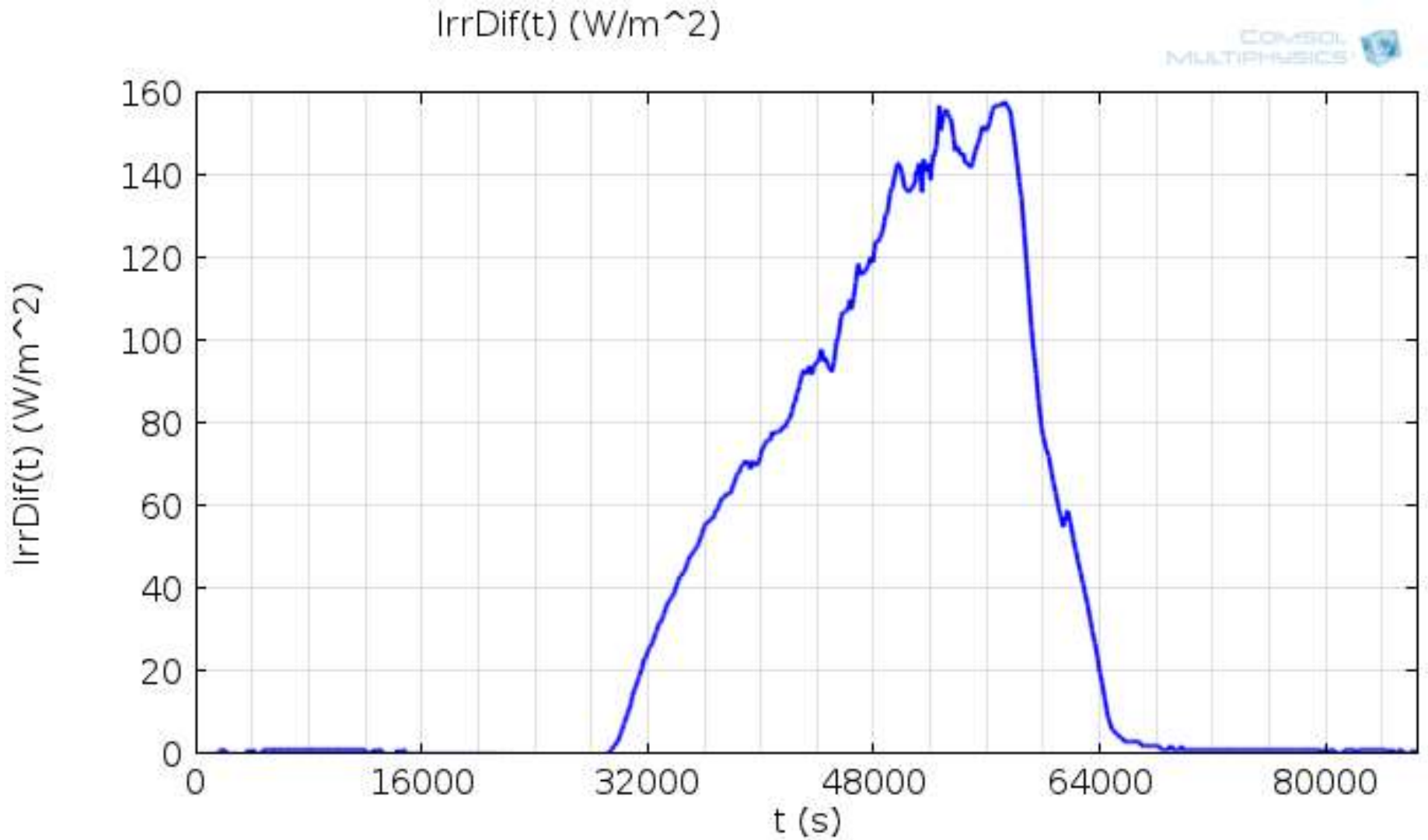


□ The full day time-averaged wind speed is 2.19 m/s.

10 Irradiation and meteorological measurements



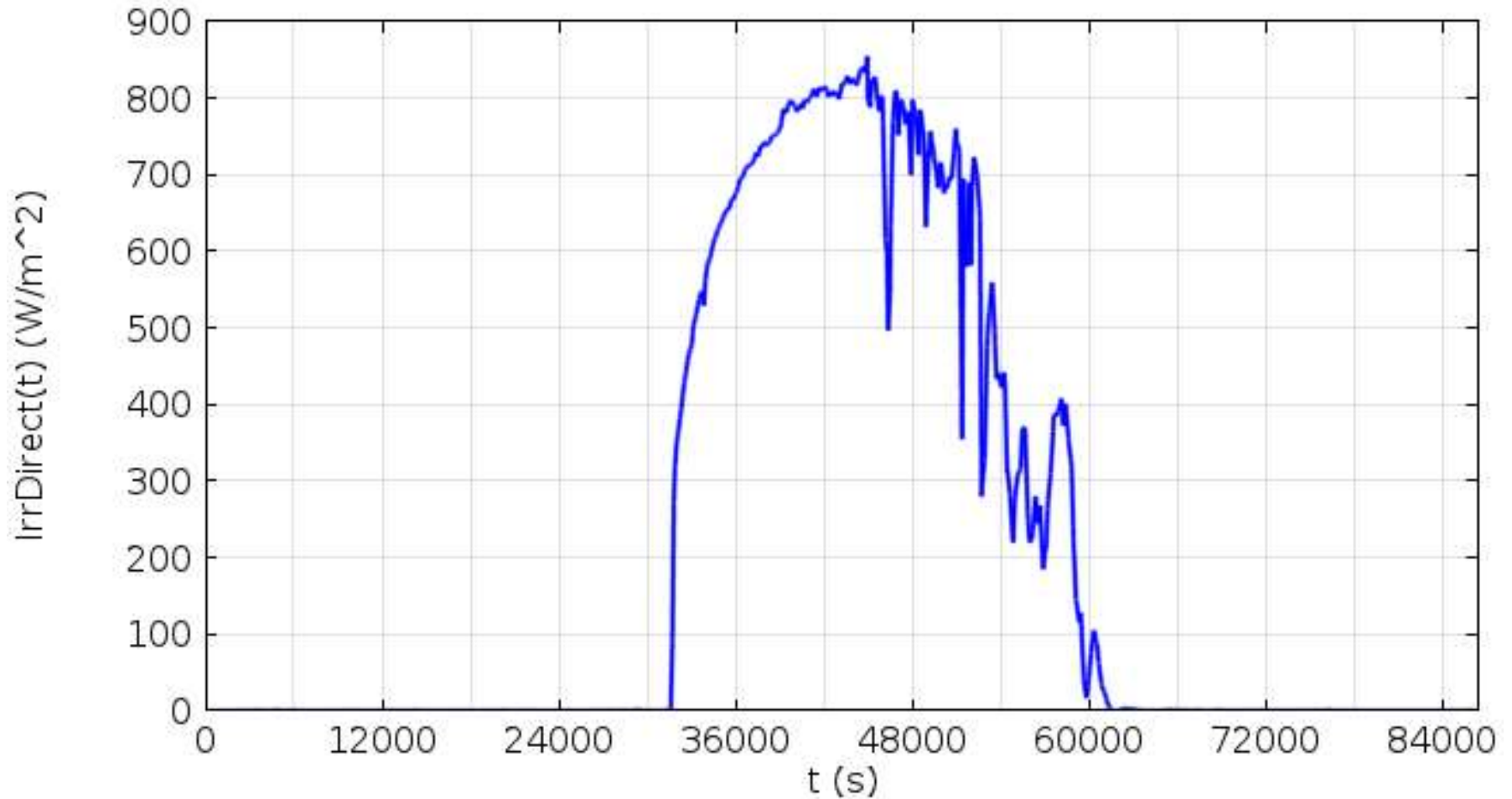
11 Irradiation and meteorological measurements



12 Irradiation and meteorological measurements

IrrDirect(t) (W/m²)

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Thermal equations

- ❑ We use the **Heat Transfer in Fluids** interface.
- ❑ Radiation is neglected in this initial studies.
- ❑ **GASEOUS SUBDOMAIN:** Heat conduction and convection,

$$\rho C_p \frac{\partial T}{\partial t} + \rho C_p \vec{u} \cdot \vec{\nabla} T = \vec{\nabla} \cdot (k \vec{\nabla} T)$$

- ❑ The velocity field is obtained from the Navier-Stokes equations that are solved in the **Laminar Flow** interface.
- ❑ The density and thermal conductivity of the air are related with the temperature.

- ▲  Heat Transfer in Fluids (ht)
 -  Heat Transfer in Fluids 1
 -  Thermal Insulation 1
 -  Initial Values 1
 -  Symmetry 1
 -  Temperature 1
 -  Boundary Heat Source 1
 -  Outflow 1
 -  Thin Thermally Resistive Layer 1










Thermal equations

□ **PV SOLID INTERIOR BOUNDARY:** Thin thermally resistive layer (4 layers) and boundary heat generation,

$$-\vec{n} \cdot (k\vec{\nabla}T) = Q_b$$

□ Q_b has a value that corresponds to the instantaneous incident direct irradiation plus the diffuse radiation.

Material	Thickness	Thermal conductivity
Glass	3 mm	1.7 W/(m·K)
Silicon	0.4 mm	0.235 W/(m·K)
EVA	0.8 mm	148 W/(m·K)
Tedlar	0.05 mm	0.158 W/(m·K)

- ▲  Heat Transfer in Fluids (ht)
 -  Heat Transfer in Fluids 1
 -  Thermal Insulation 1
 -  Initial Values 1
 -  Symmetry 1
 -  Temperature 1
 -  Boundary Heat Source 1
 -  Outflow 1
 -  Thin Thermally Resistive Layer 1

Fluid equations

□ They are the corresponding to the **Turbulent Flow, k-ε** interface.

□ We have wind conditions and, in this initial calculation, we neglect the **body force** acting on the fluid that corresponds to the buoyancy force due to the dependence of the density of the air with the temperature.

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho(\mathbf{u} \cdot \nabla)\mathbf{u} =$$

$$\nabla \cdot \left[-p\mathbf{I} + (\mu + \mu_T)(\nabla\mathbf{u} + (\nabla\mathbf{u})^T) - \frac{2}{3}(\mu + \mu_T)(\nabla \cdot \mathbf{u})\mathbf{I} - \frac{2}{3}\rho k\mathbf{I} \right] + \mathbf{F}$$

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho\mathbf{u}) = 0$$


$$\rho \frac{\partial k}{\partial t} + \rho(\mathbf{u} \cdot \nabla)k = \nabla \cdot \left[\left(\mu + \frac{\mu_T}{\sigma_k} \right) \nabla k \right] + P_k - \rho \epsilon$$

$$\rho \frac{\partial \epsilon}{\partial t} + \rho(\mathbf{u} \cdot \nabla)\epsilon = \nabla \cdot \left[\left(\mu + \frac{\mu_T}{\sigma_\epsilon} \right) \nabla \epsilon \right] + C_{\epsilon 1} \frac{\epsilon}{k} P_k - C_{\epsilon 2} \rho \frac{\epsilon^2}{k}, \quad \epsilon = \epsilon_p$$

$$\mu_T = \rho C_\mu \frac{k^2}{\epsilon}$$

$$P_k = \mu_T \left[\nabla \mathbf{u} : (\nabla \mathbf{u} + (\nabla \mathbf{u})^T) - \frac{2}{3}(\nabla \cdot \mathbf{u})^2 \right] - \frac{2}{3}\rho k \nabla \cdot \mathbf{u}$$

▲  * Turbulent Flow, k-ε 2 (spf2)


 Fluid Properties 1

 Wall 1

 Initial Values 1

 Symmetry 1

 Interior Wall 1

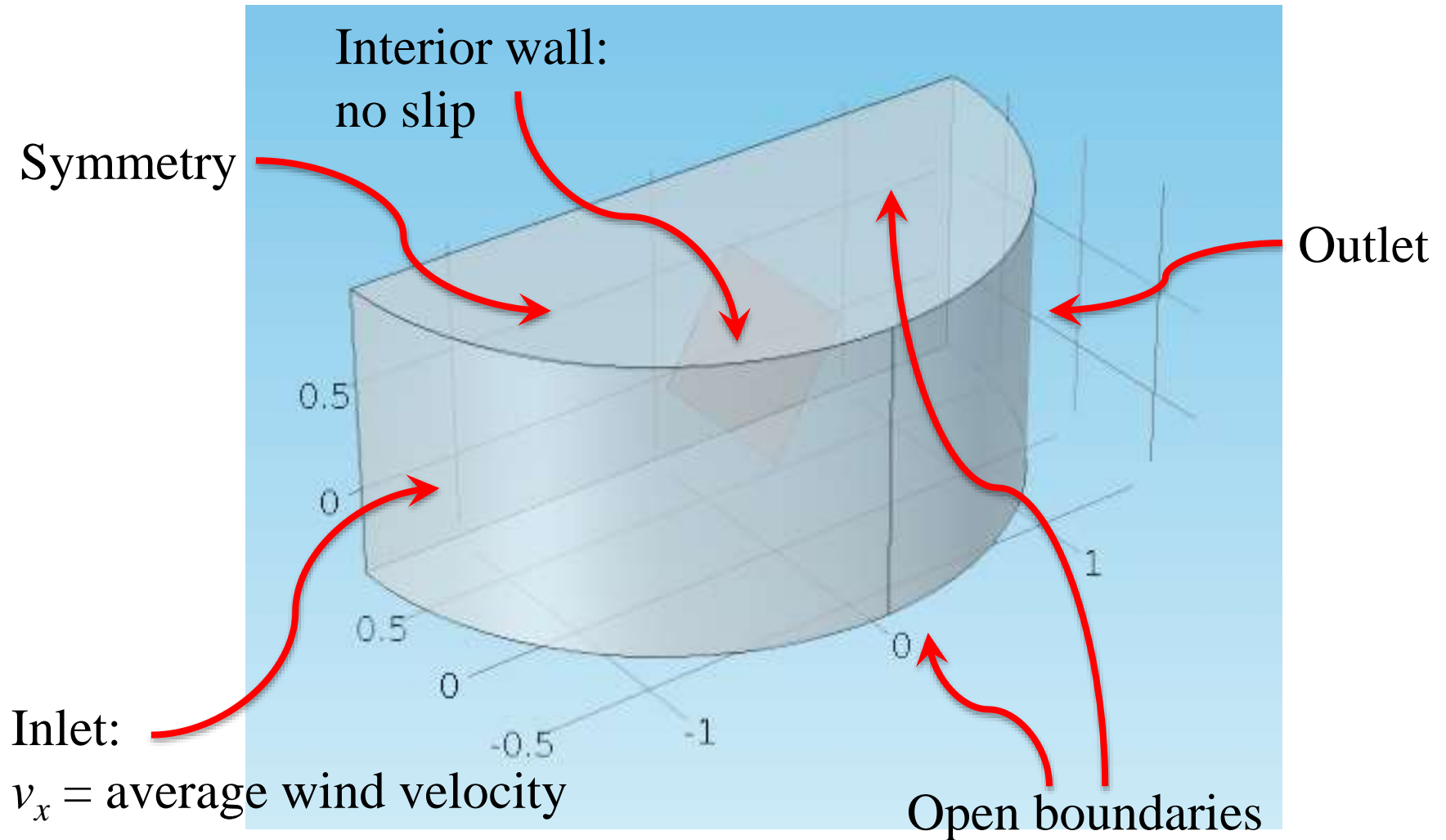
 Outlet 1

 Inlet 1

 Open Boundary 1

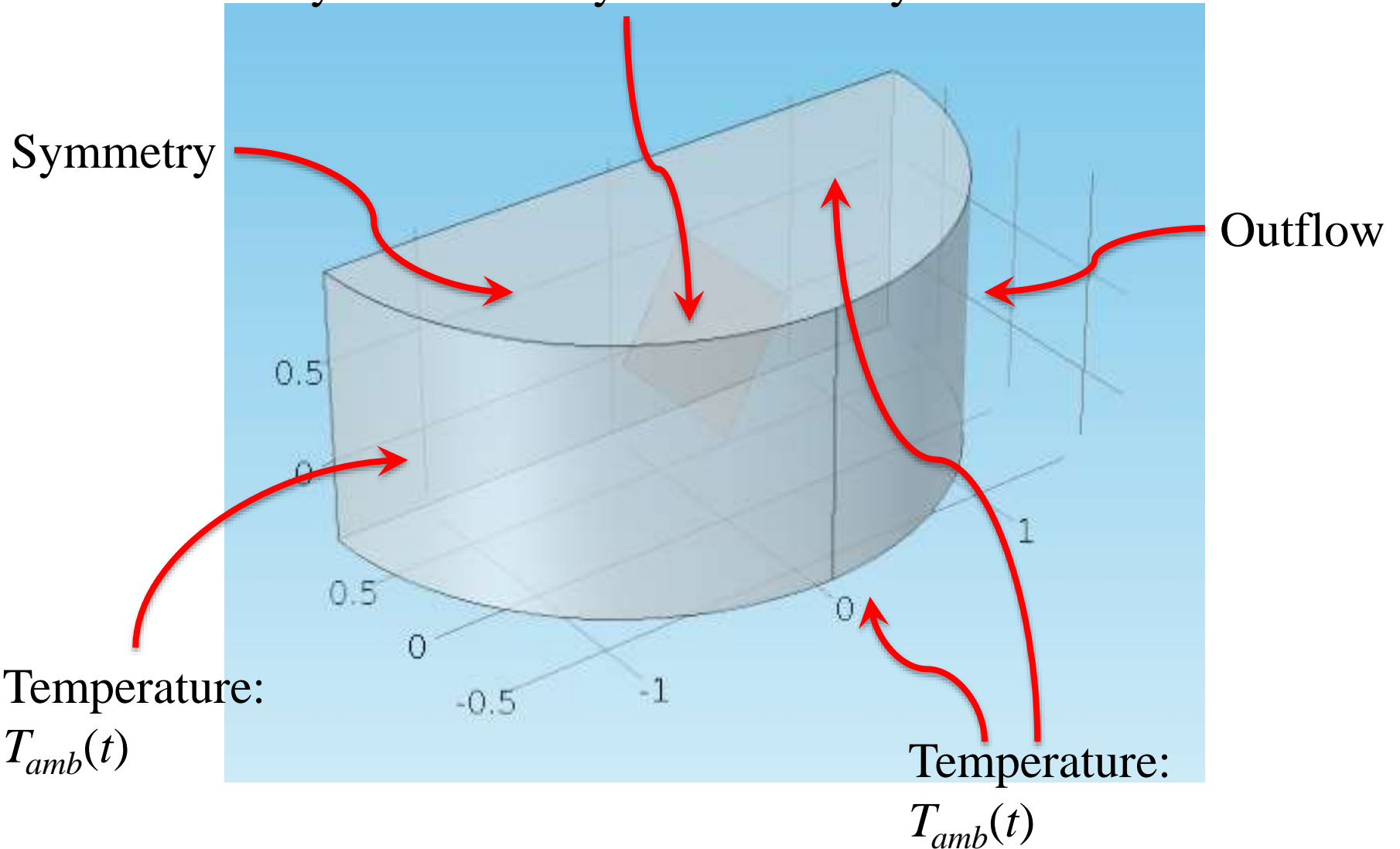
Re is approximately 60000

Fluid Boundary Conditions

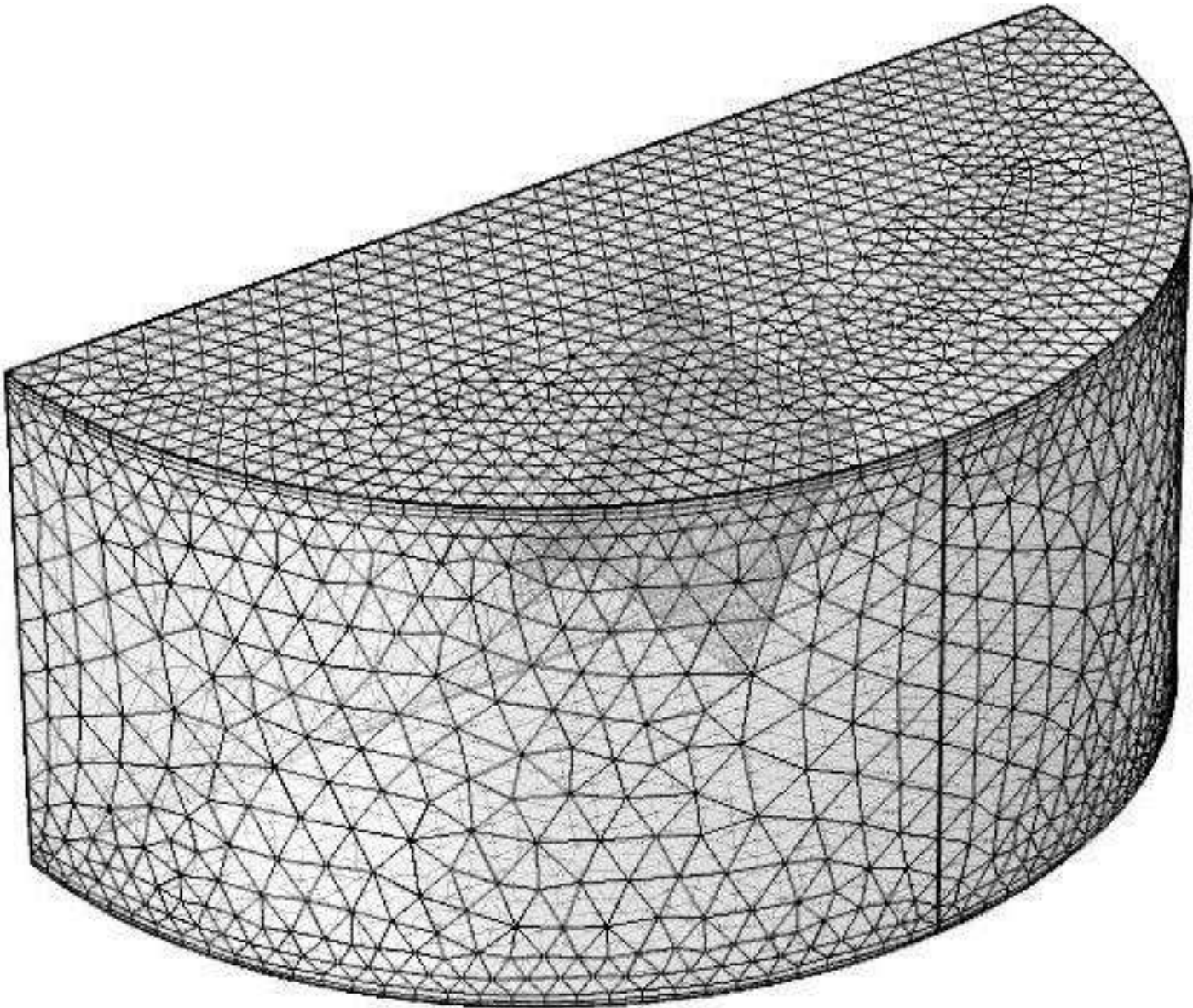


Thermal Boundary Conditions

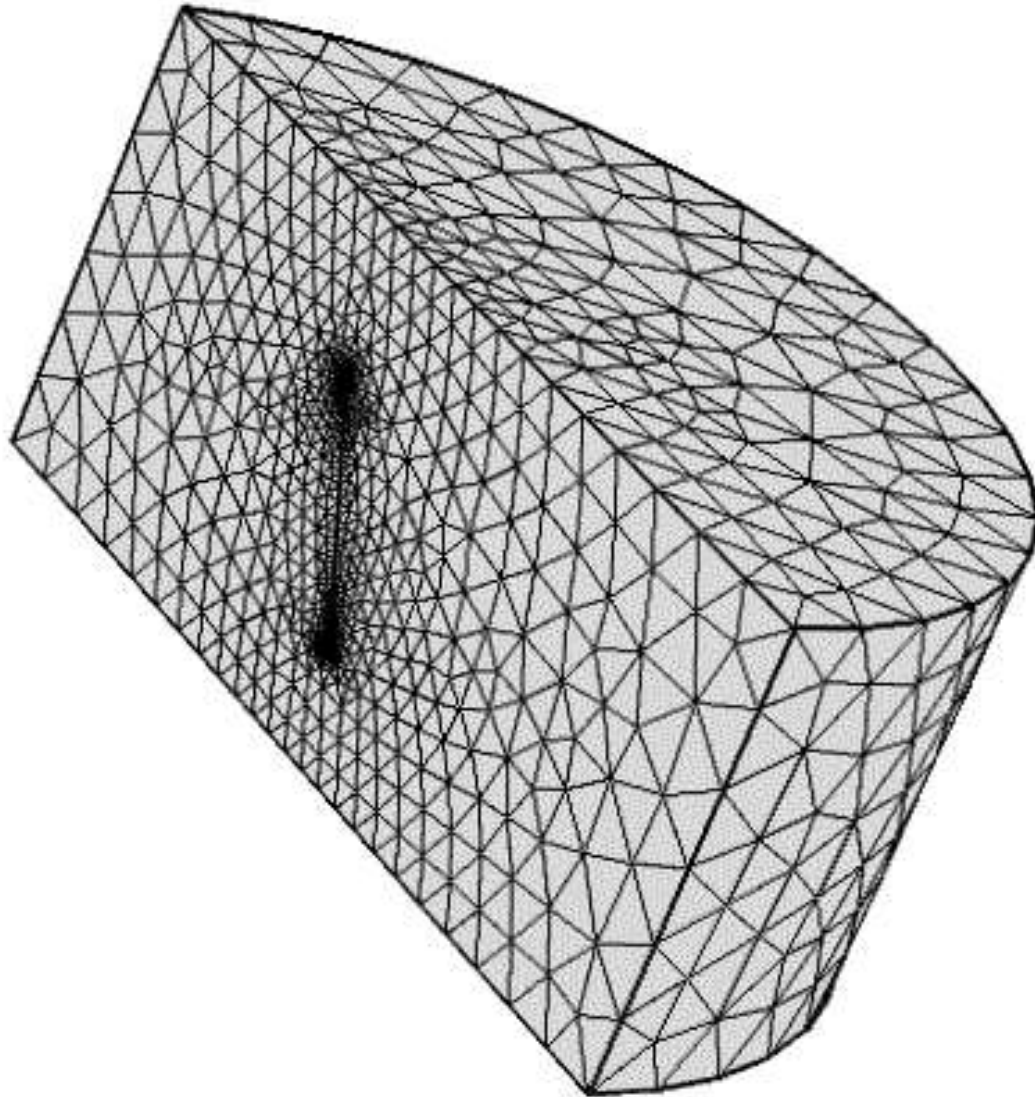
Thin Thermally Resistive Layer & Boundary Heat Source



Mesh 1



Mesh 2



Solution

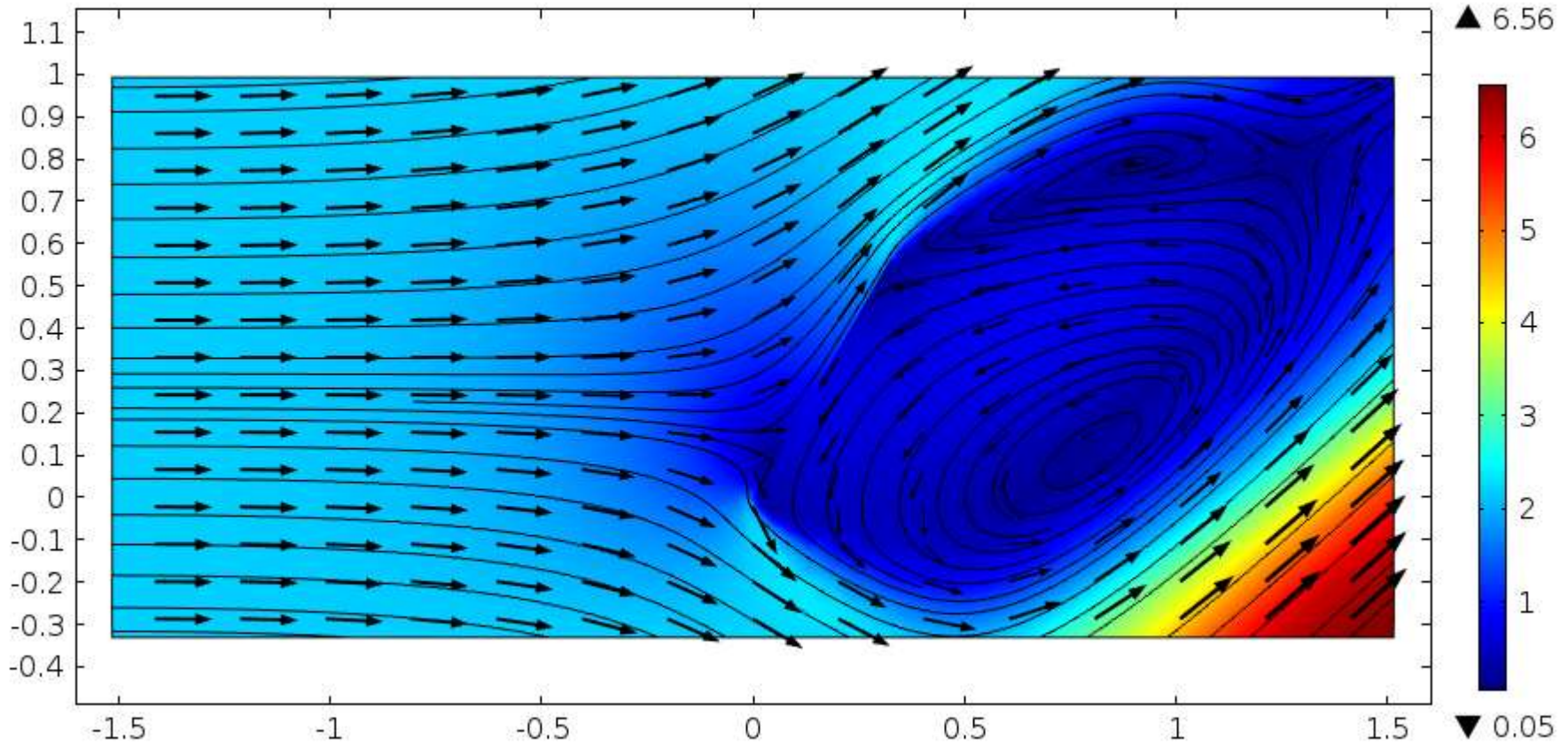
- ❑ We have a one-direction coupling system.

- ❑ We use two study time-dependent steps:
 - ❑ In the ***first step***, we calculate the air velocity field under the averaged inlet wind velocity, neglecting the dependence of the air viscosity and density with the temperature.
 - ❑ In the ***second step***, we solve for the temperature distribution, by using the velocity field calculated in the previous step.

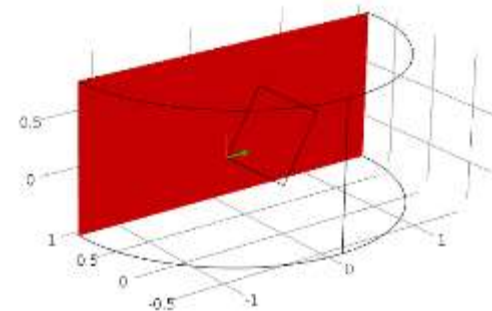
Results

Surface: Velocity magnitude (m/s) Streamline: Velocity field Arrow Surface: Velocity field

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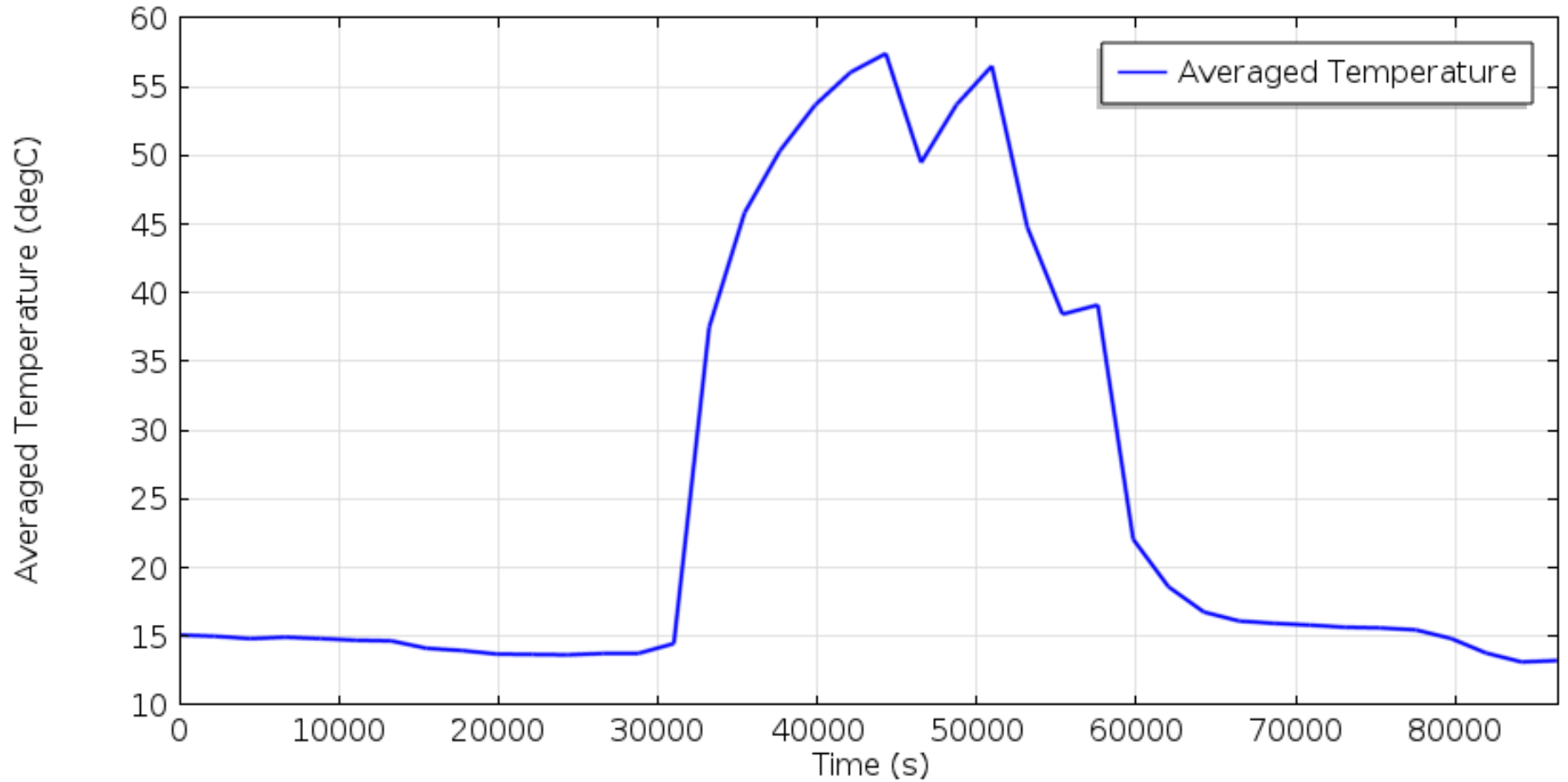
Mid plane velocity field.



Results

Global: Averaged Temperature (degC)

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Averaged temperature of the solar panel.

Results

From the calculations and measurements, we can observe that:

- ❑ The time dependent temperature of the module **agree well** with the values founded in experiments.
- ❑ The simulated temperature of the module is **a little bit higher than the measured ones**.
- ❑ **The temperature differences** from different points in the panel is about $2\text{ }^{\circ}\text{C} - 3\text{ }^{\circ}\text{C}$.
- ❑ **There are two air vortexes**, with roughly 30 cm – 40 cm diameter, downstream behind the module.

Conclusions

- ❑ We have shown the **use of COMSOL Multiphysics** in the thermal simulation of a photovoltaic module.
- ❑ From the numerical results, **we can extract information** about the temperature fields and heat transfer with the ambient medium.
- ❑ In particular, this procedure **could be applied in the determination of the heat coefficients of photovoltaic modules** made with different technologies, under several installation conditions and wind speeds.

Future work

- ❑ We have started with a very simple simulation and we now have to:
 - ❑ Include the effect of the time-dependent wind velocity.
 - ❑ Take into account the variable wind direction.
 - ❑ Make the strong coupling.
 - ❑ Introduce the more complex geometry of the solar tracker system.

Bibliography

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2. K. Emery, ***Measurement and Characterization of Solar Cells and Modules***, in *Handbook of Photovoltaic Science and Technology*, A. Luque and S. Hegedus, eds. (2003).
3. M. C. Alonso García and J. L. Balenzategui, *Renewable Energy*, **29**, 1997-2010 (2004).

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Thank you very much for your attention!



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