

Heat Output from Space Heating Radiator with Add-on-fan Blowers

Per-Olof Johansson
Janusz Wollerstrand

Dept. for Energy sciences
Lund University



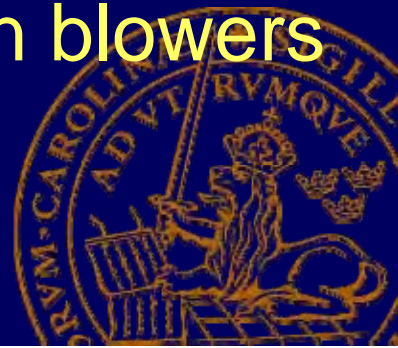
Background

DH temperatures are dependent on the temperature demand in the connected buildings

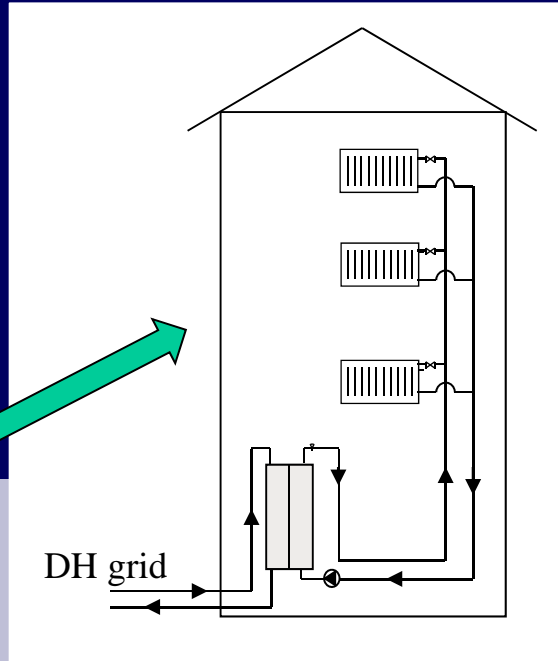
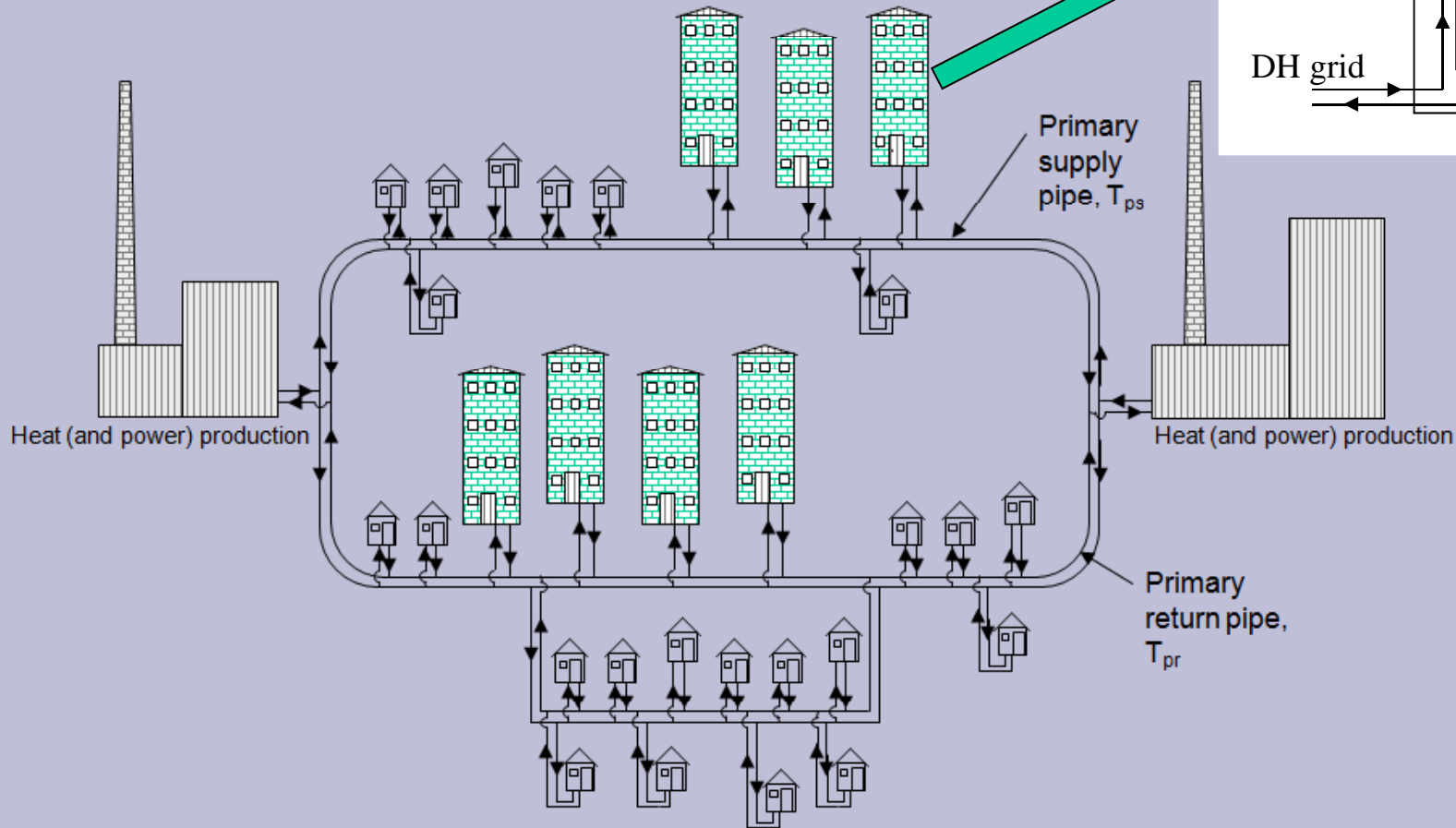
Space heating return temperature describes the District Heating (DH) return temperature

Low temperature heating systems leads the way towards a reduced DH temperature

Improved heat output without changing existing space heating system by using add-on-fan blowers

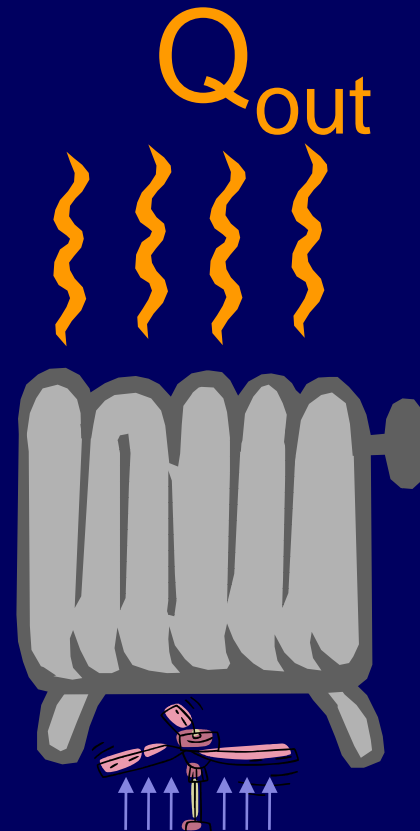
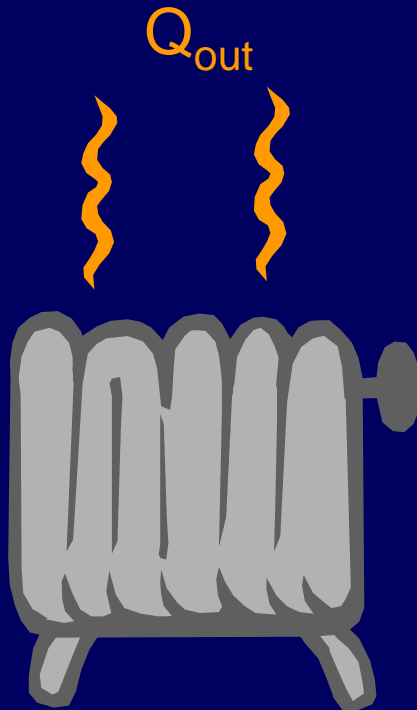


District Heating



Add-on-fan blower

Increased forced convection to increase the heat output from the space heating system



Objective

Simulate the affect of add-on-fan blowers at different fan speeds

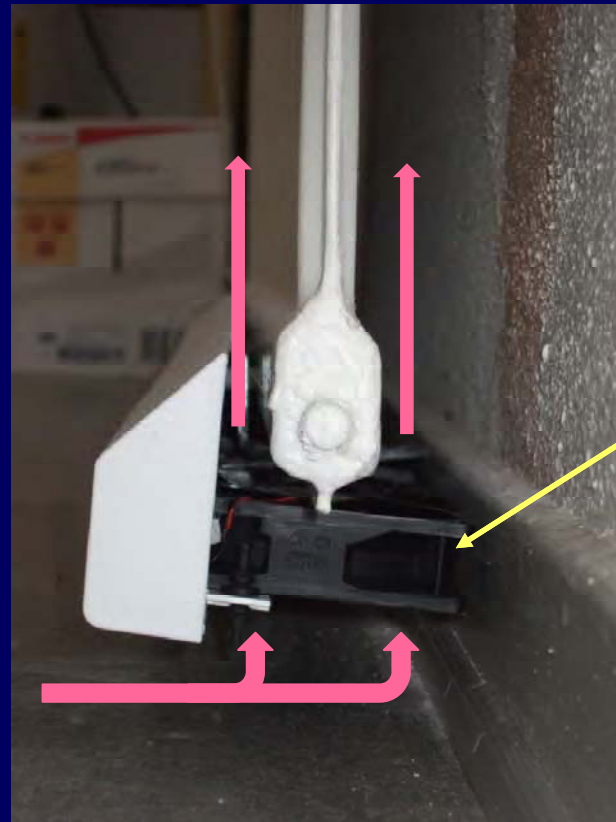
Derive new space heating temperature program

Complement to a field study



Add-on-fan blower – Field study

Test setup

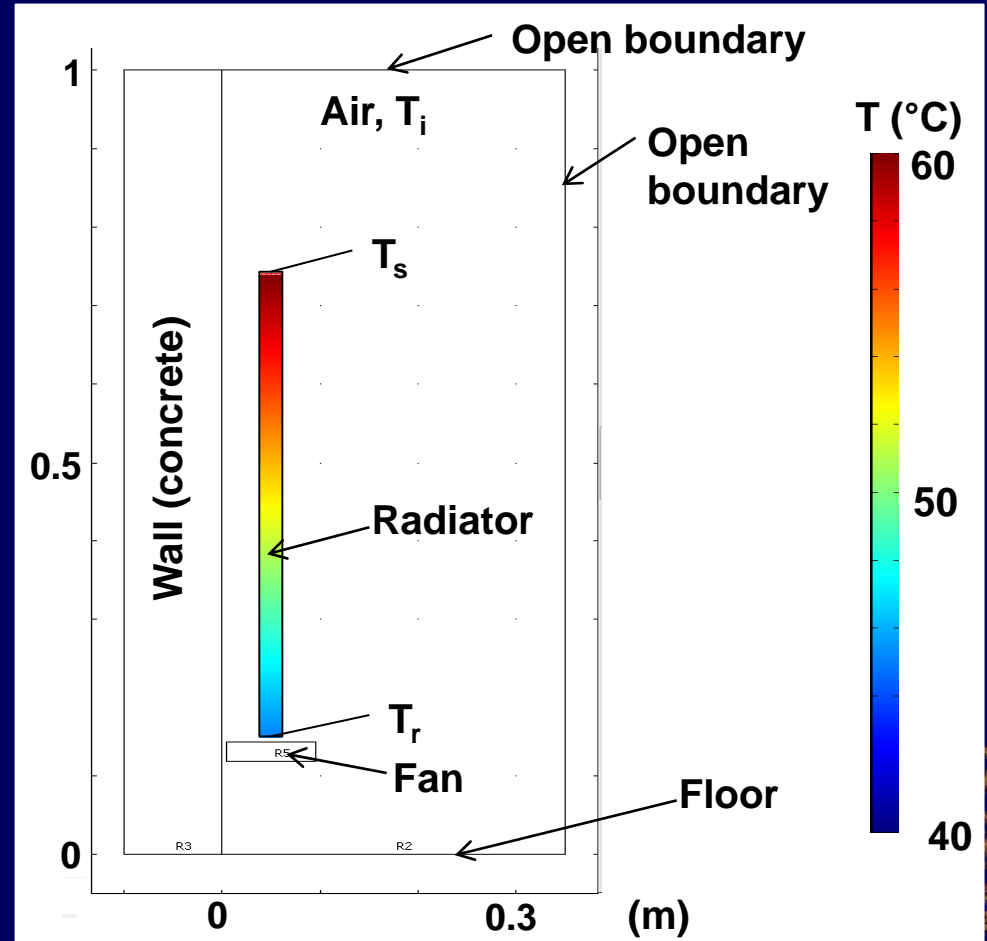
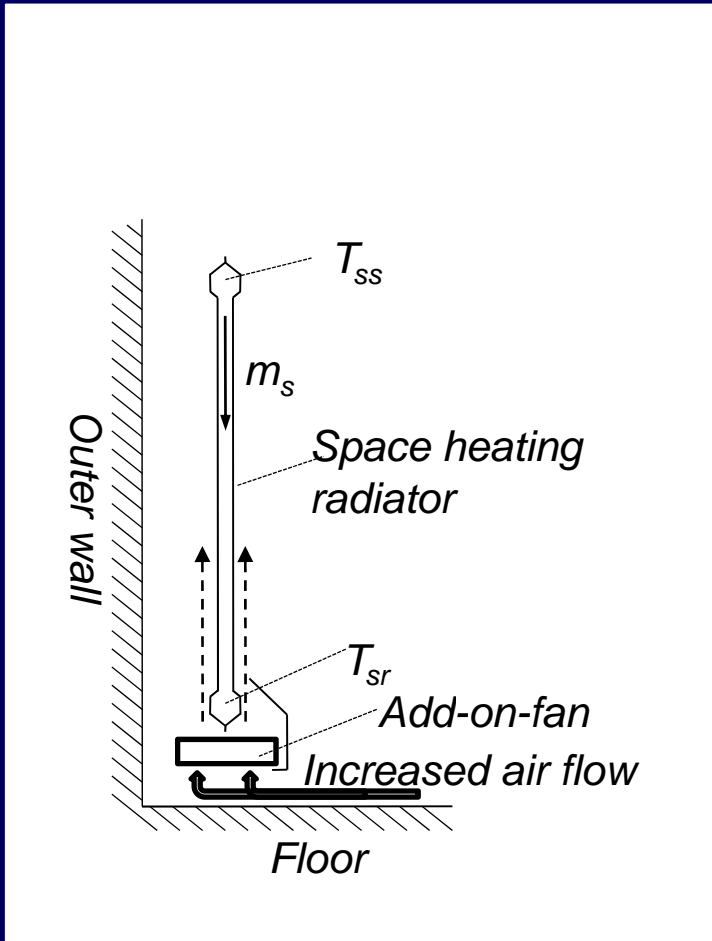


Additional add-on-fans

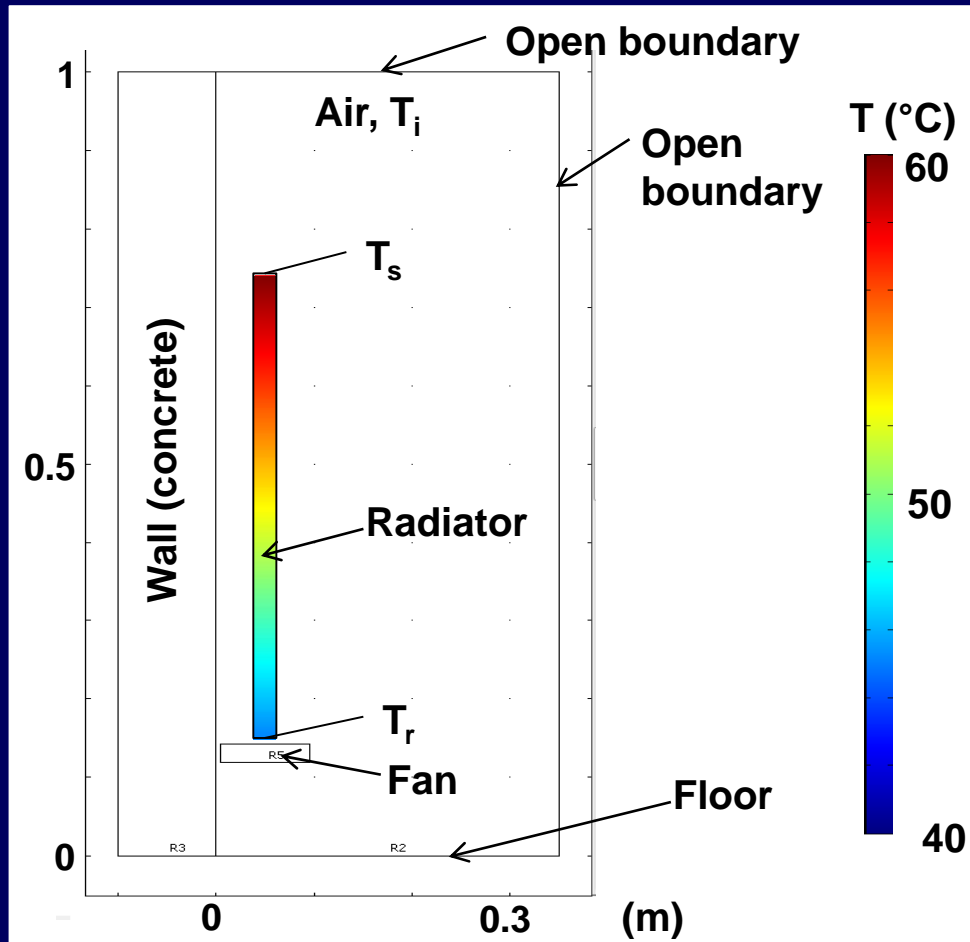


Add-on-fan blower – Simulation

COMSOL 2-D model



COMSOL-model



Limitations and assumptions:

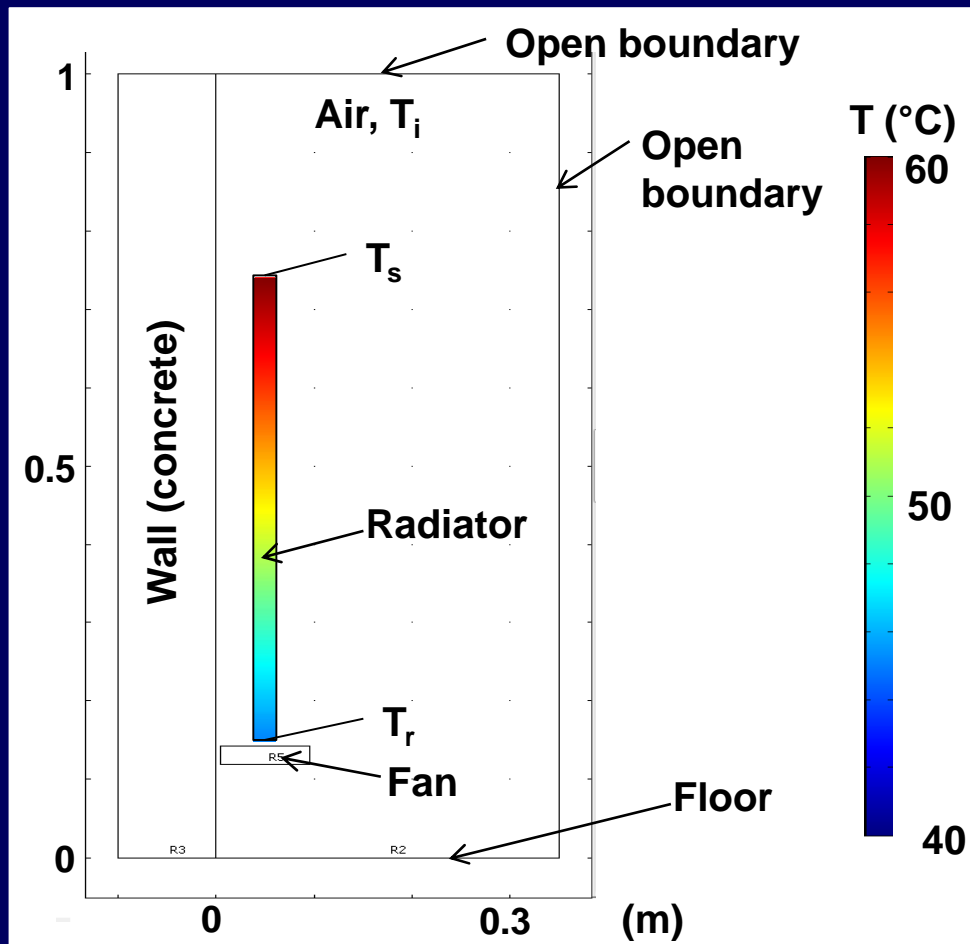
No water flow in the radiator

A linear temperature in the middle of the radiator is assumed

No heat losses in the outer wall



COMSOL-model



The model is using:

General Heat Transfer toolbox

Navier-Stokes toolbox (Weakly Compressible and Incompressible)

The model is dealing with:

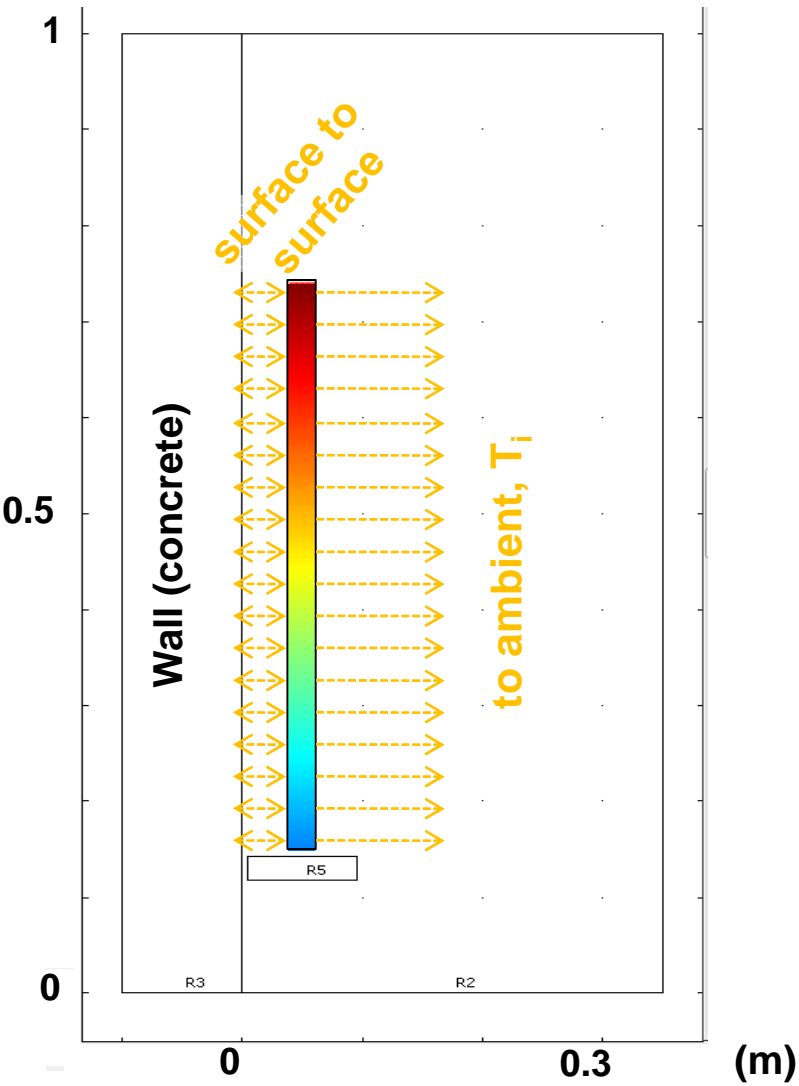
Convection (natural and forced)

Radiation

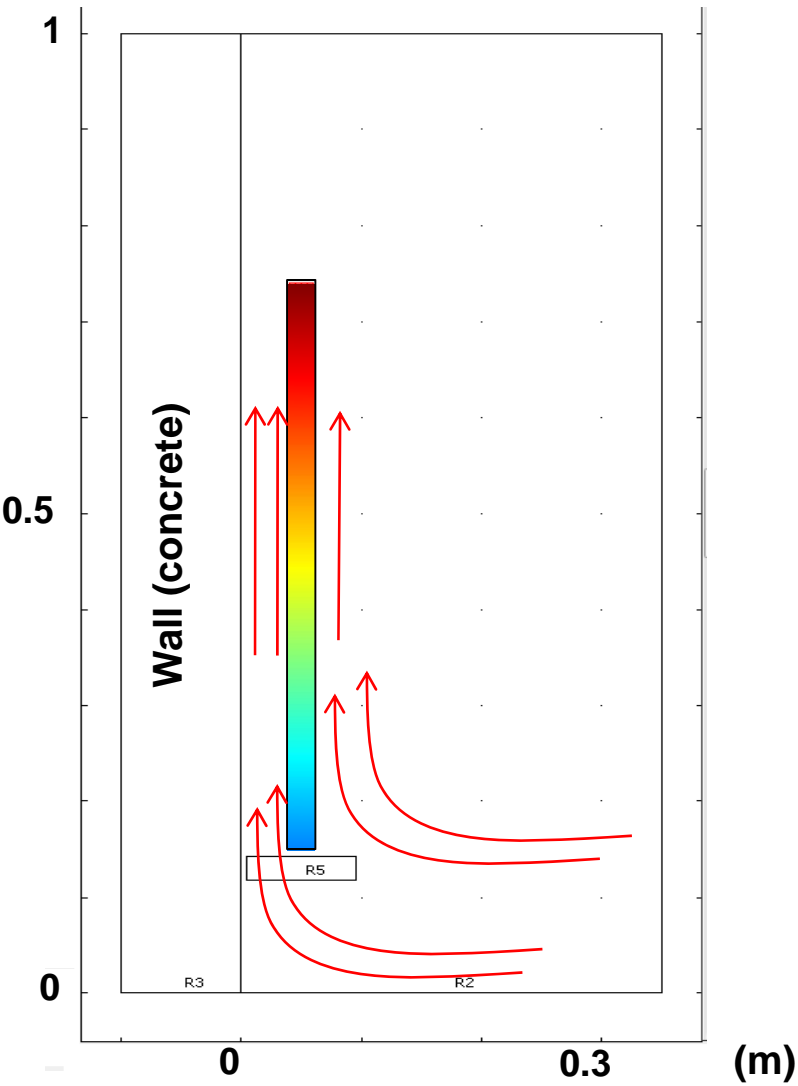


COMSOL-model

RADIATION

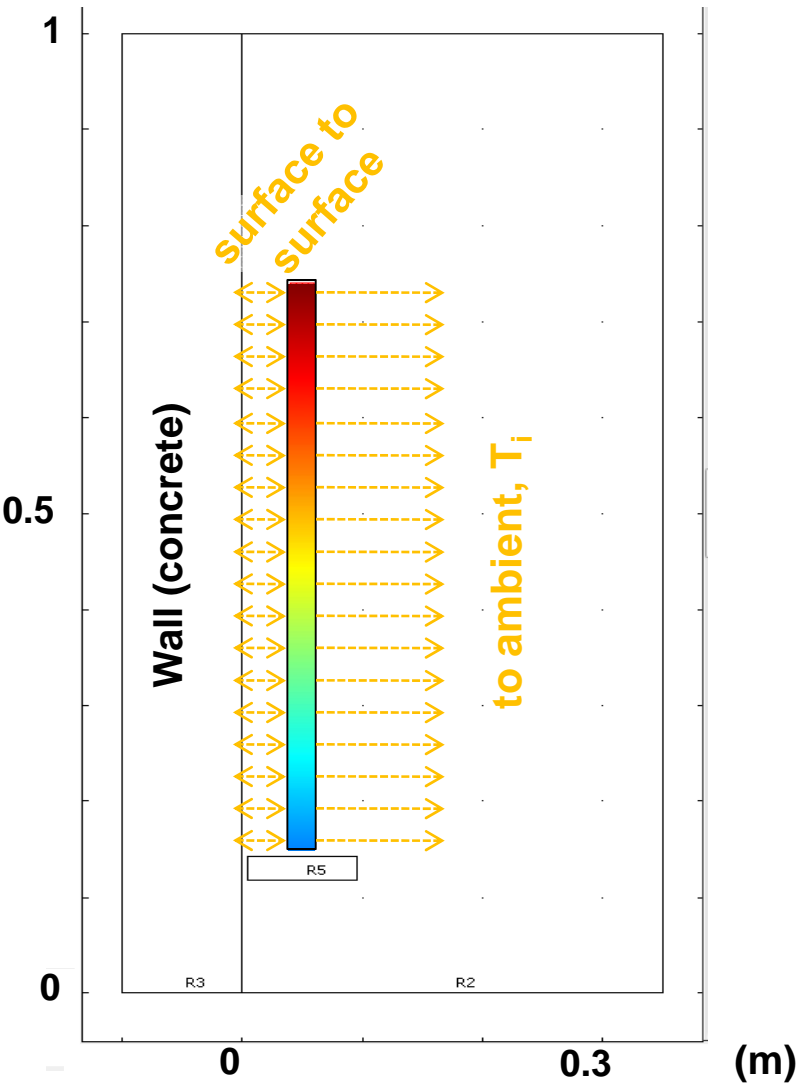


NATURAL CONVECTION

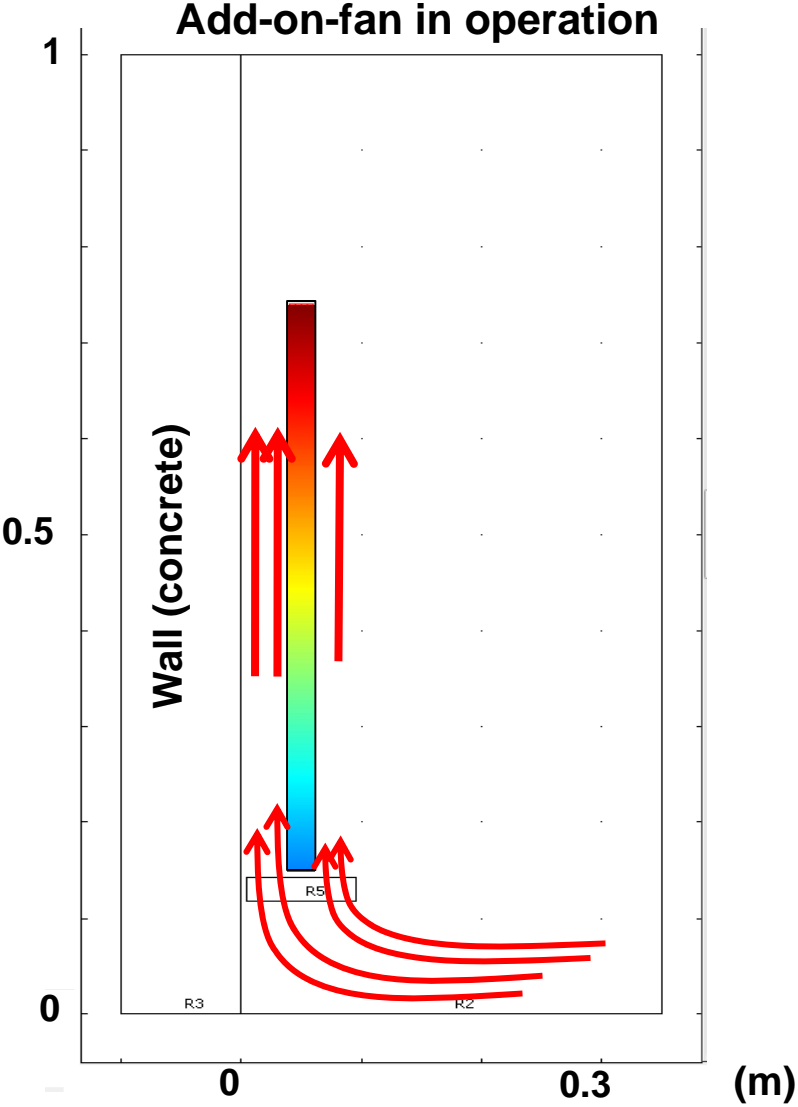


COMSOL-model

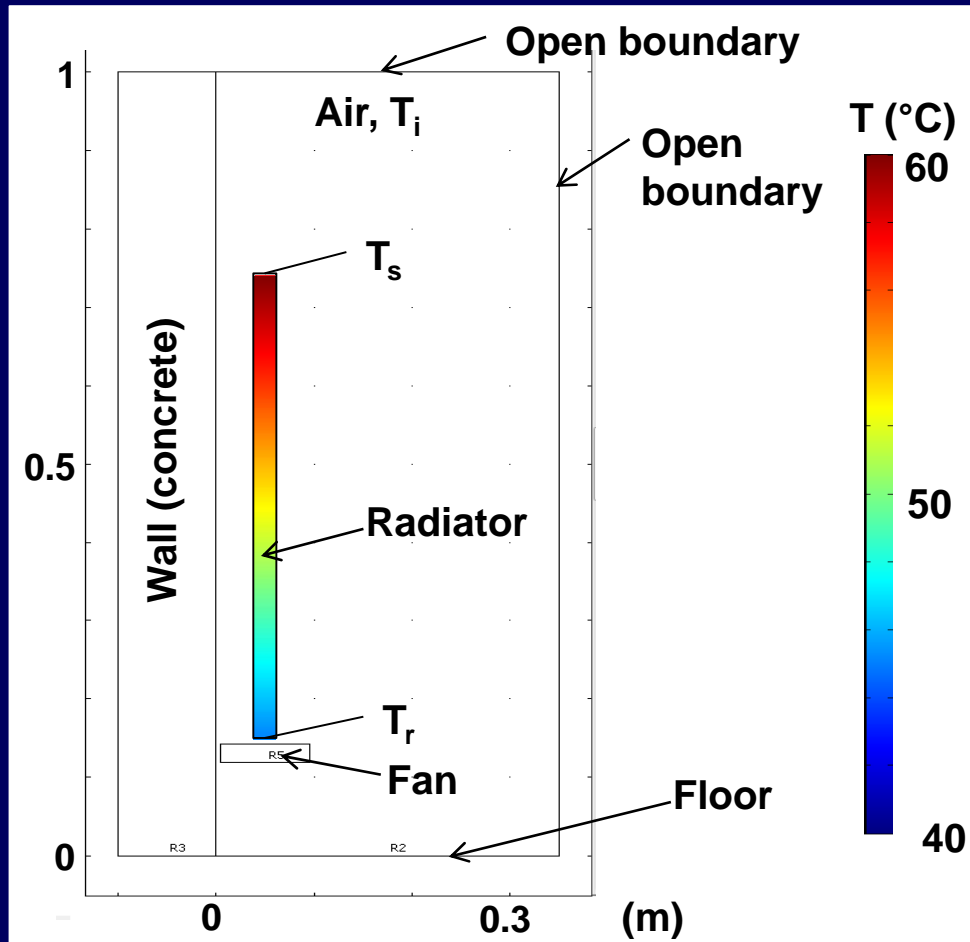
RADIATION



FORCED CONVECTION



COMSOL model



The model is validated for

Natural convection

Forced convection

Parameter test:

Mesh size

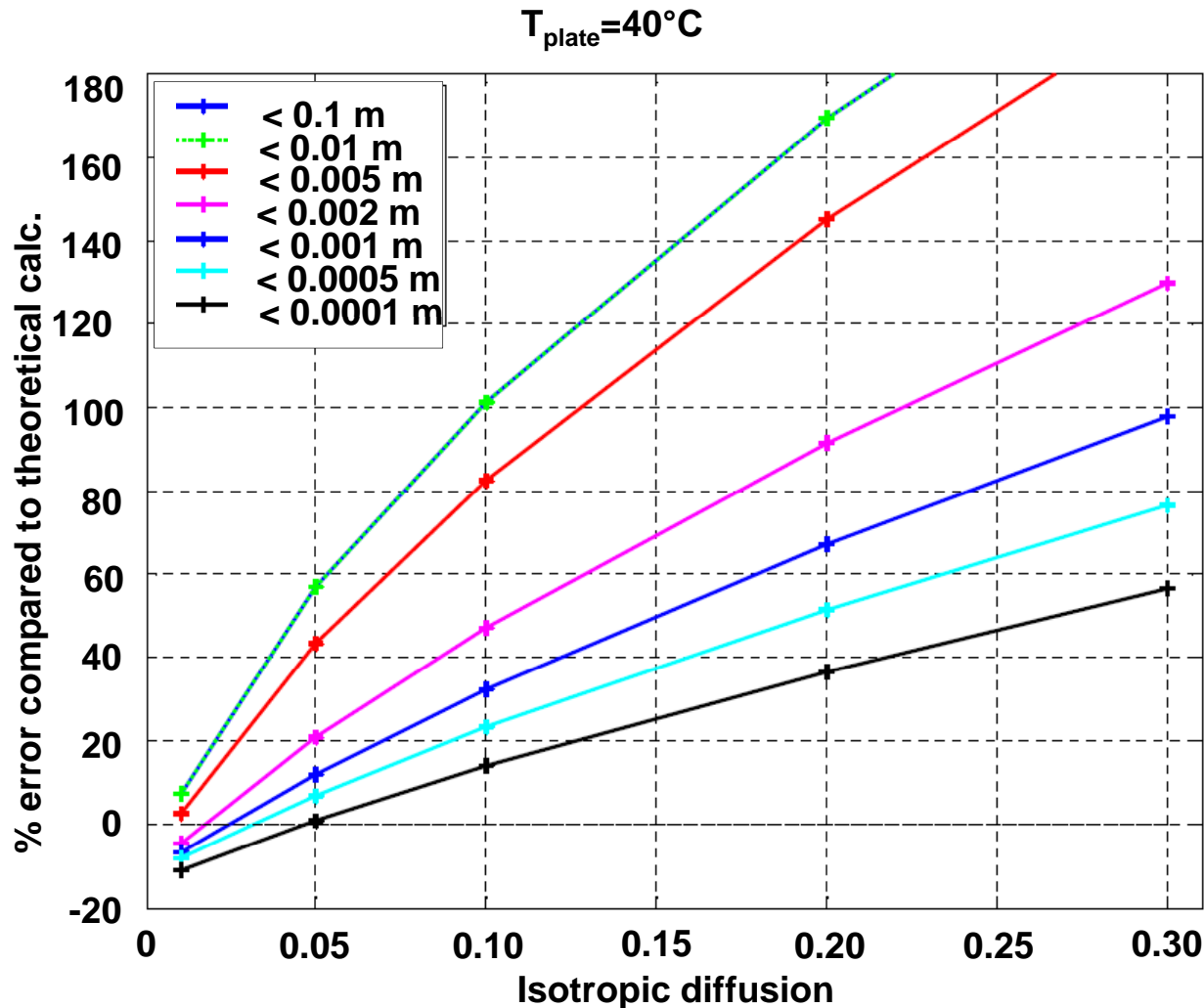
Isotropic diffusion



COMSOL-model

- validation

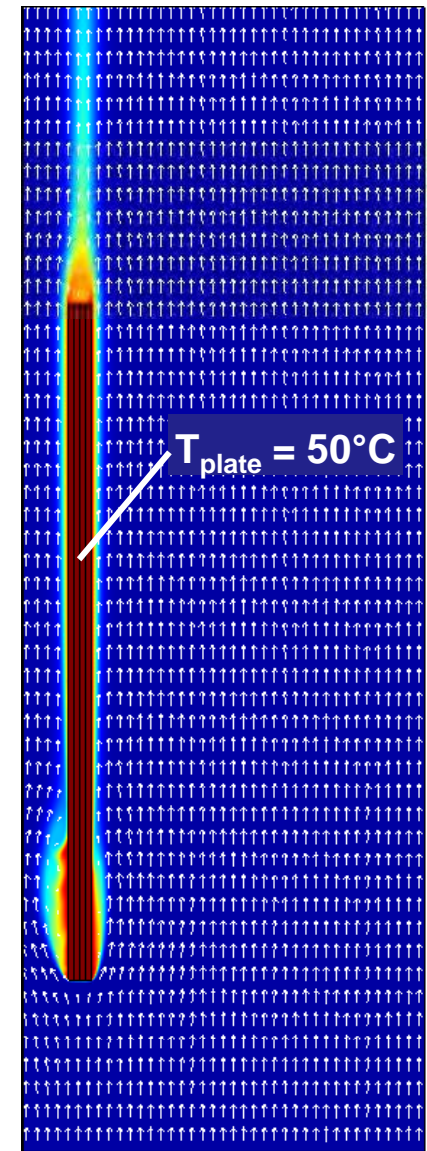
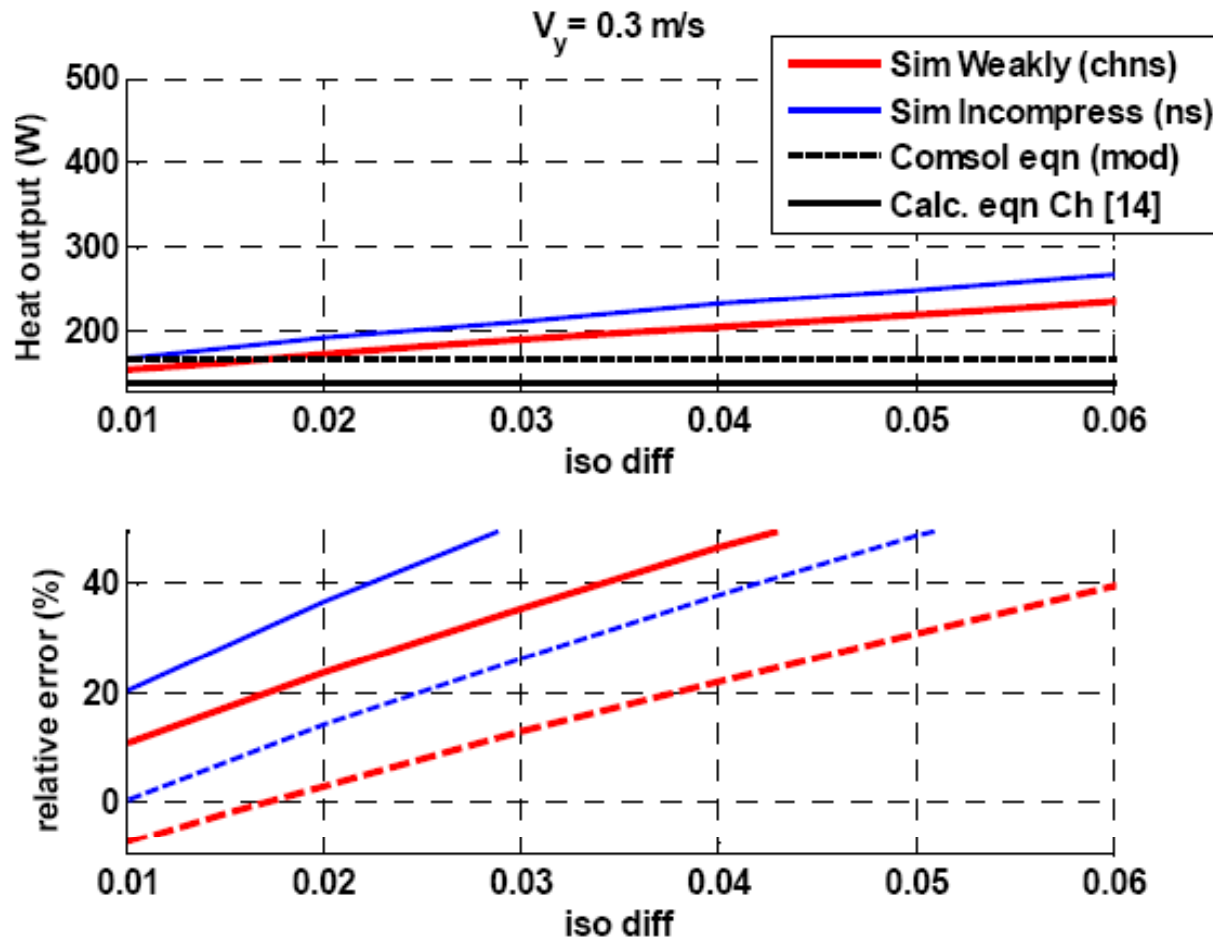
Natural convection



COMSOL-model

- validation

Forced convection



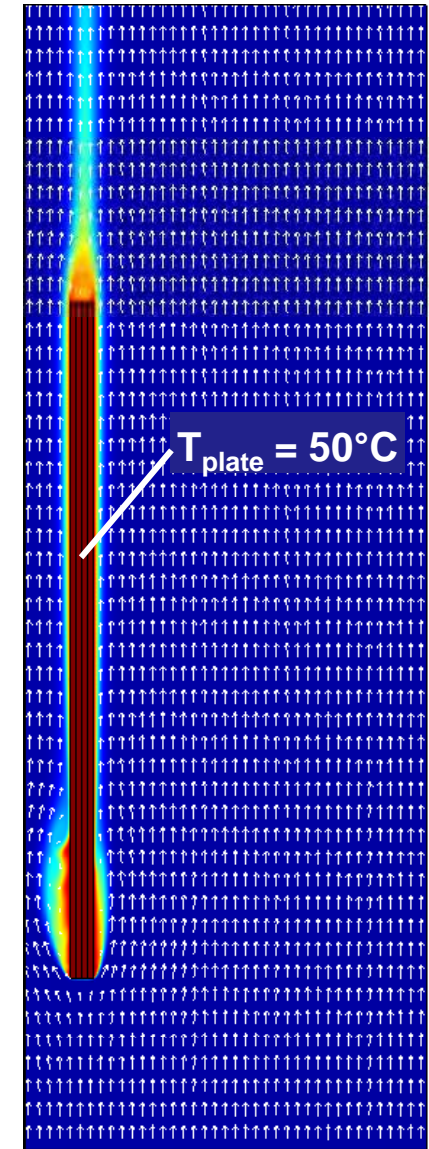
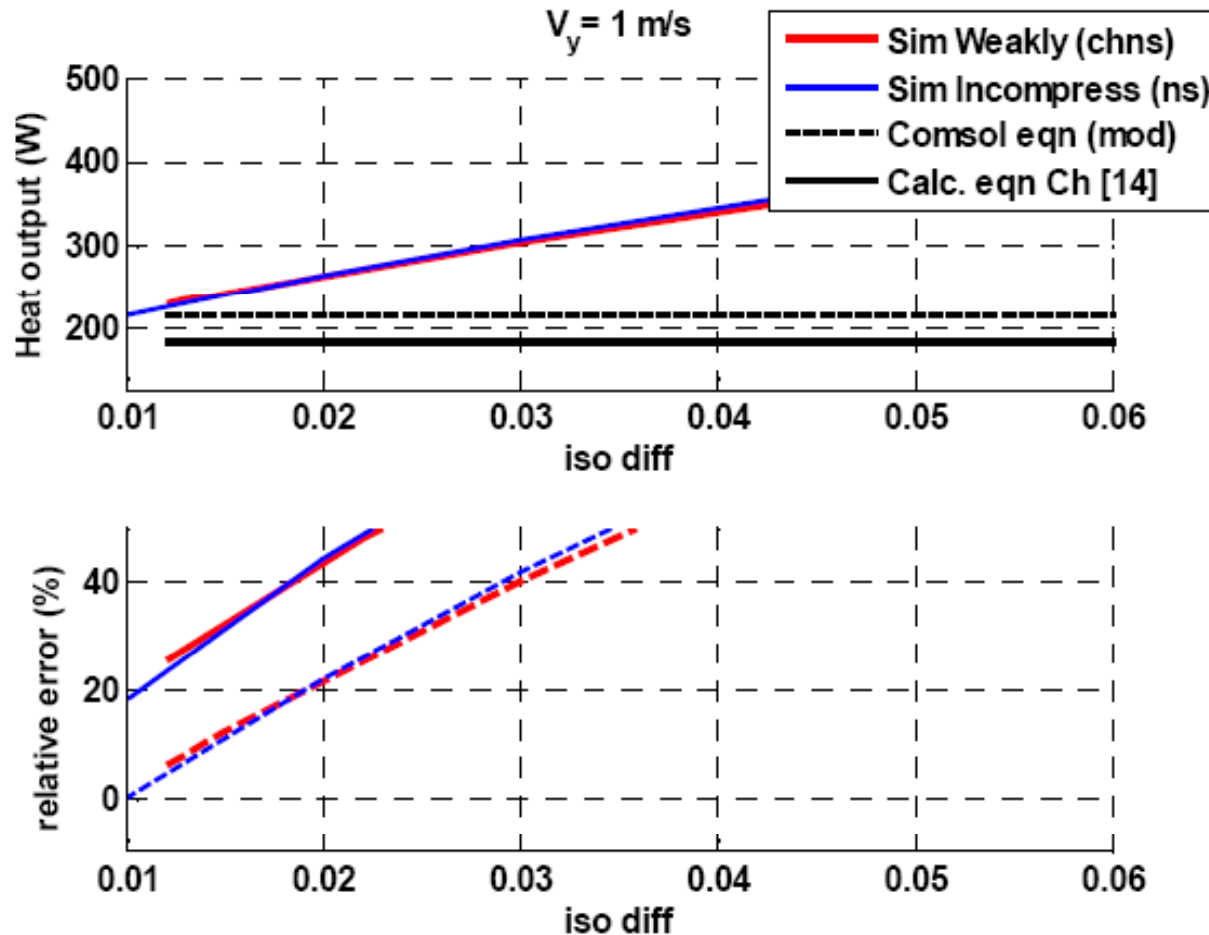
Constant velocity, inlet



COMSOL-model

- validation

Forced convection



Constant velocity, inlet



COMSOL-model

- Deriving new temperature program

To derive new temperature program the heat output should be the same as in the reference case without add-on-fan blower

$$Q = \dot{m}_s \cdot c_p (T_s - T_r) = k \cdot A \cdot \Delta\theta$$

Two control strategies is considered:

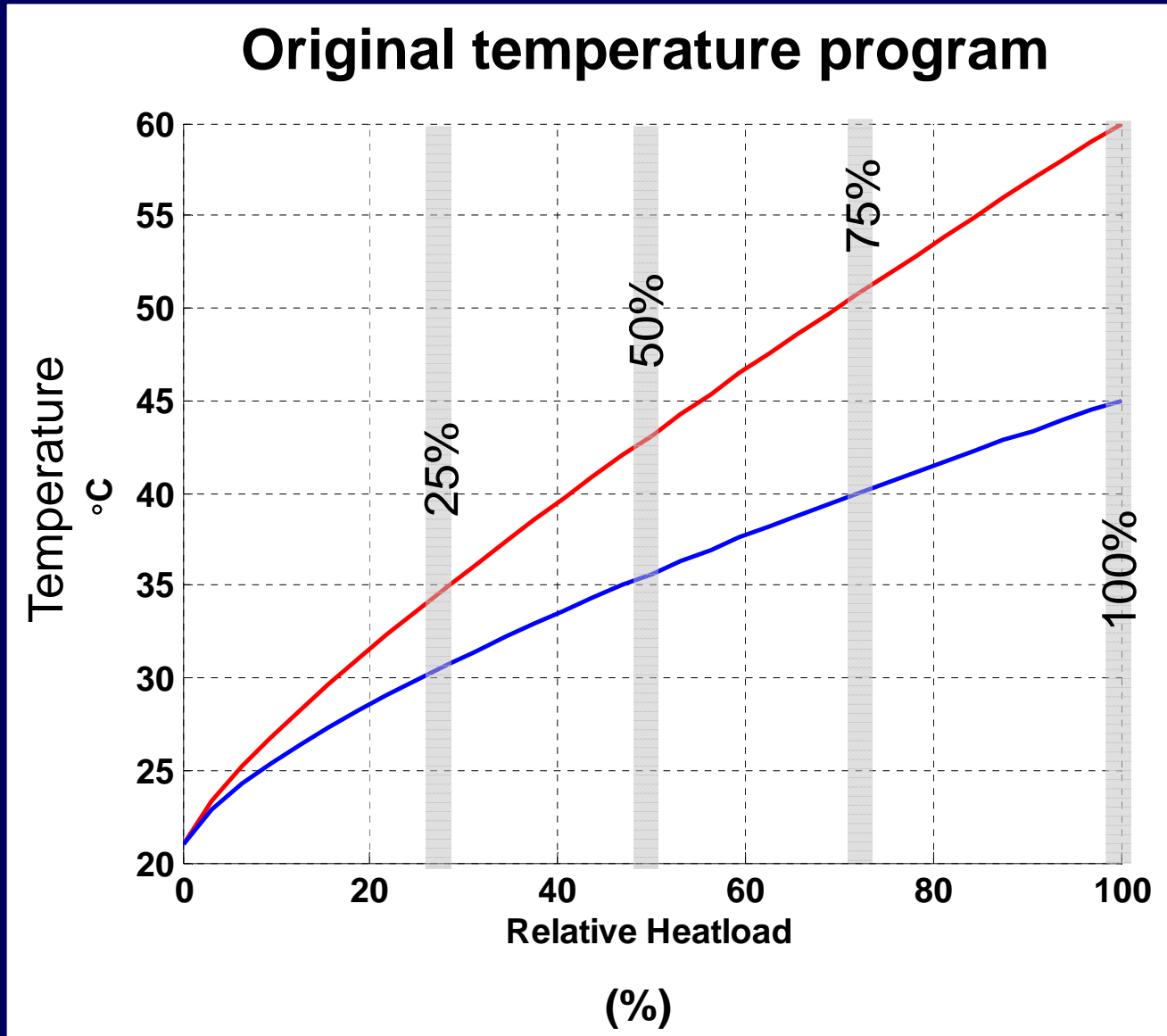
1. Constant mass flow (m_s) through the radiator $\rightarrow T_s$ is reduced and $(T_s - T_r)$ is constant
2. Constant $T_s \rightarrow m_s$ is reducing and $(T_s - T_r)$ is increasing

$$\Delta\theta = \frac{T_s - T_r}{\ln \frac{T_s - T_i}{T_r - T_i}}$$



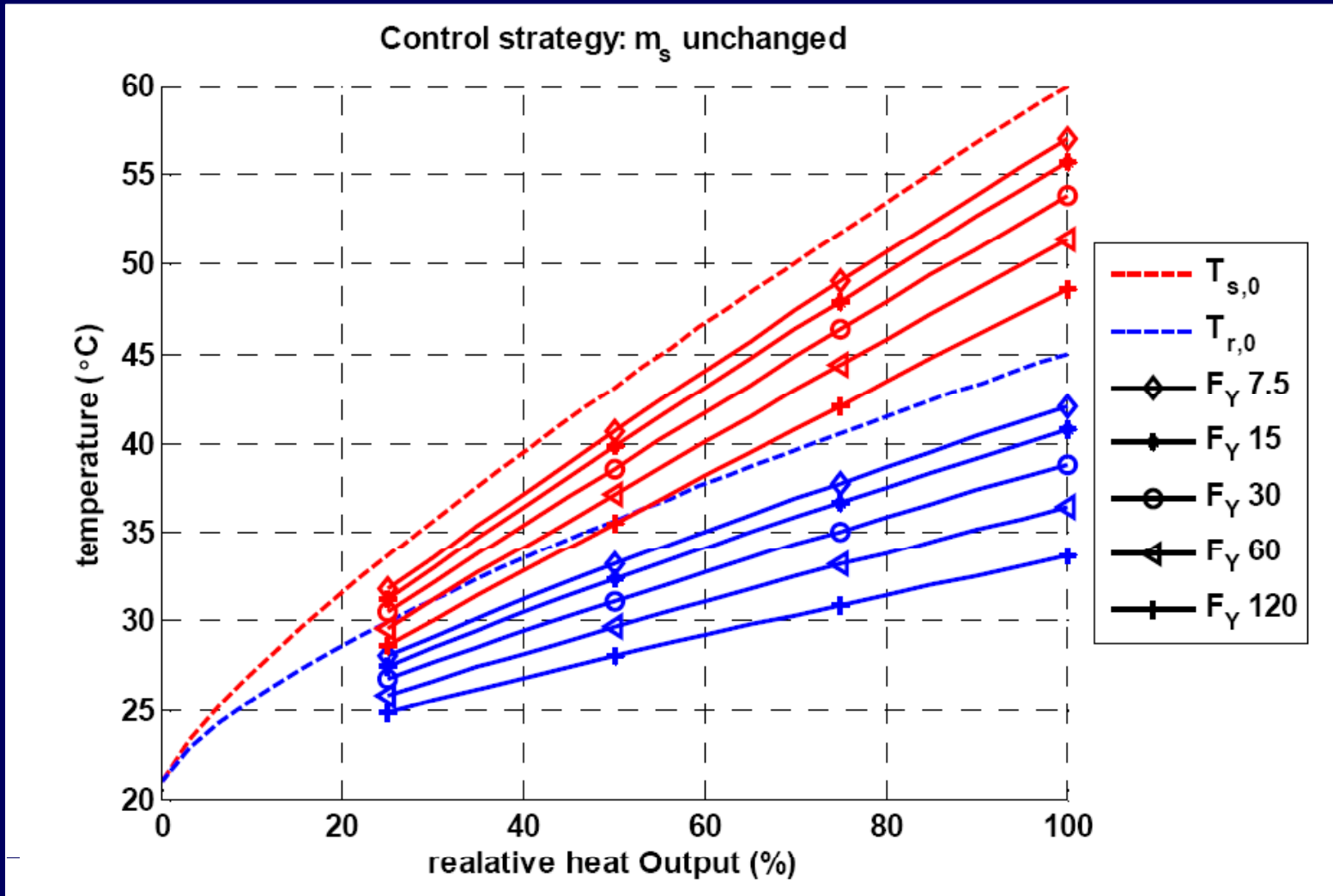
COMSOL-model

- Deriving new temperature program



COMSOL-model

- Deriving new temperature program, Results



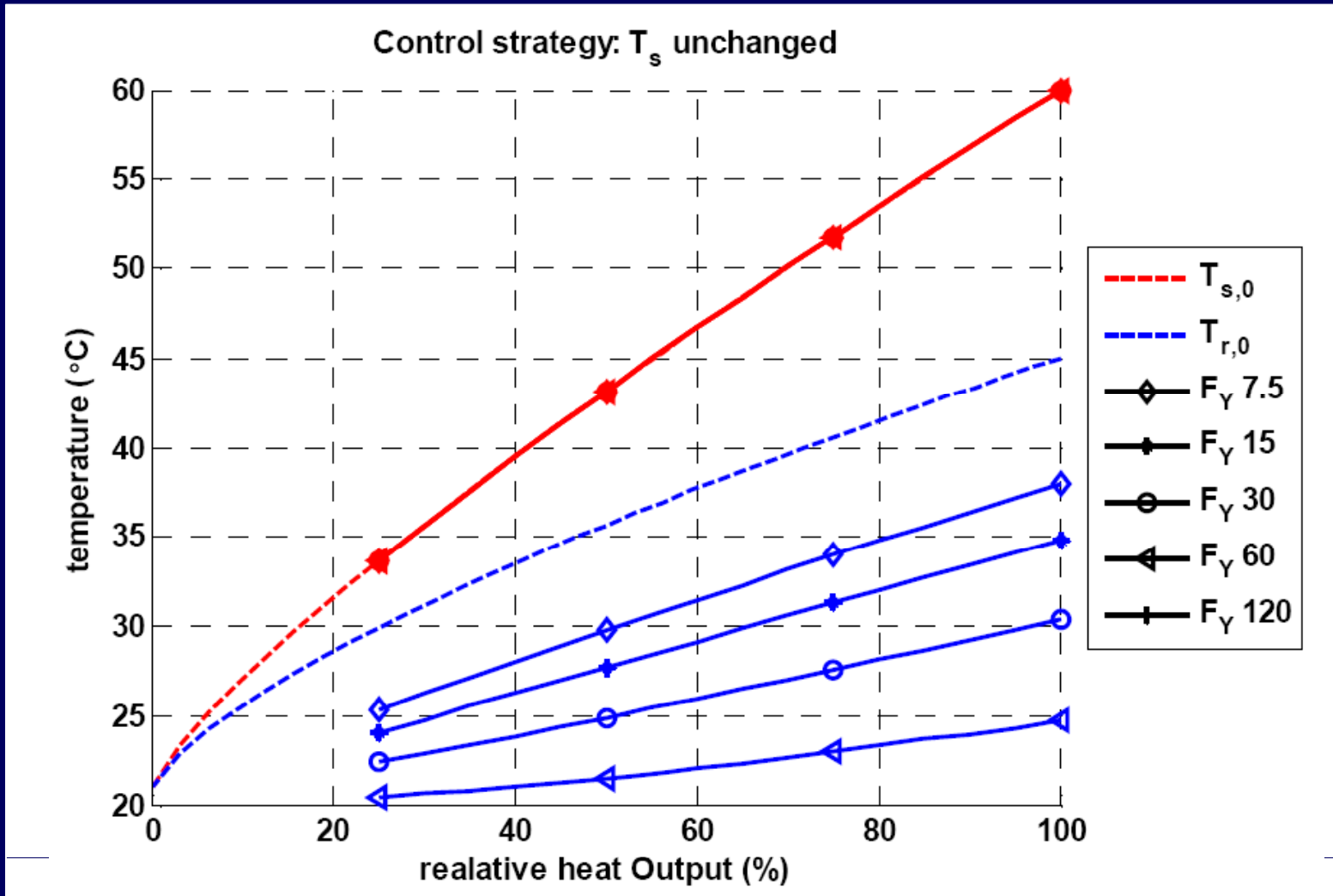
SOLVER: Time dependent Direct UMFPACK

Simulation time: 259200 sec (=3*24h)



COMSOL-model

- Deriving new temperature program, Results



SOLVER: Time dependent Direct UMFPACK

Simulation time: 259200 sec (=3*24h)



COMSOL-model

- Deriving new temperature program, Results

Vertical force F_y (N/m ³)	Air volume flow V_{air} (m ³ /s)	Δp (N/m ²)	P_{fan} (W)	P_{fan} $\eta=0.1$ (W)
7.5	0.025	0.11	0.003	0.03
15	0.036	0.26	0.009	0.09
30	0.051	0.55	0.028	0.28
60	0.073	1.12	0.081	0.81
120	0.104	2.23	0.231	2.31



Conclusions

- A significant reduction of space heating temperature program is possible with add-on-fan blowers
- The model shows good correspondence to theoretical calculations within simulated range
- It is possible to derive new temperature program for a variation of fan speeds
- The model shows good correspondence to field study

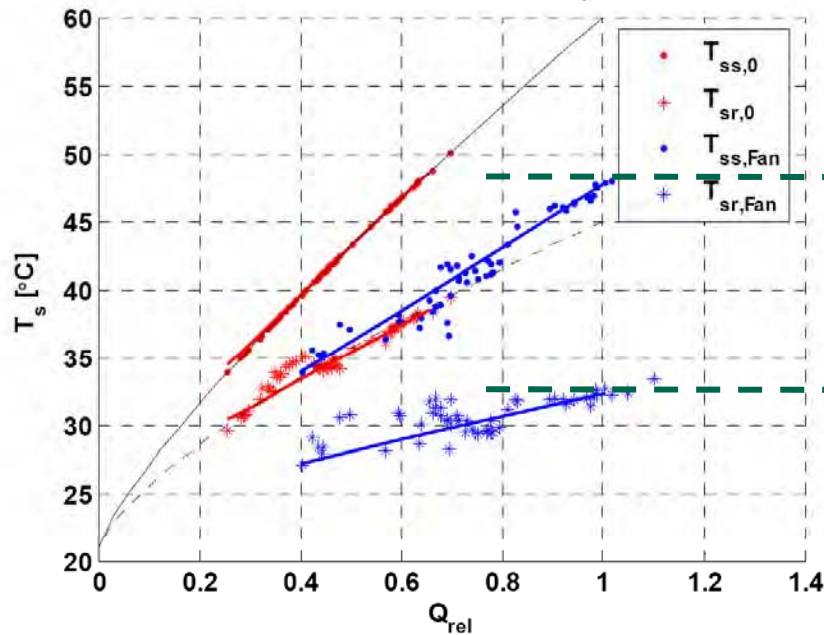


COMSOL-model

- Results, Field study V.S. Simulation

Field study

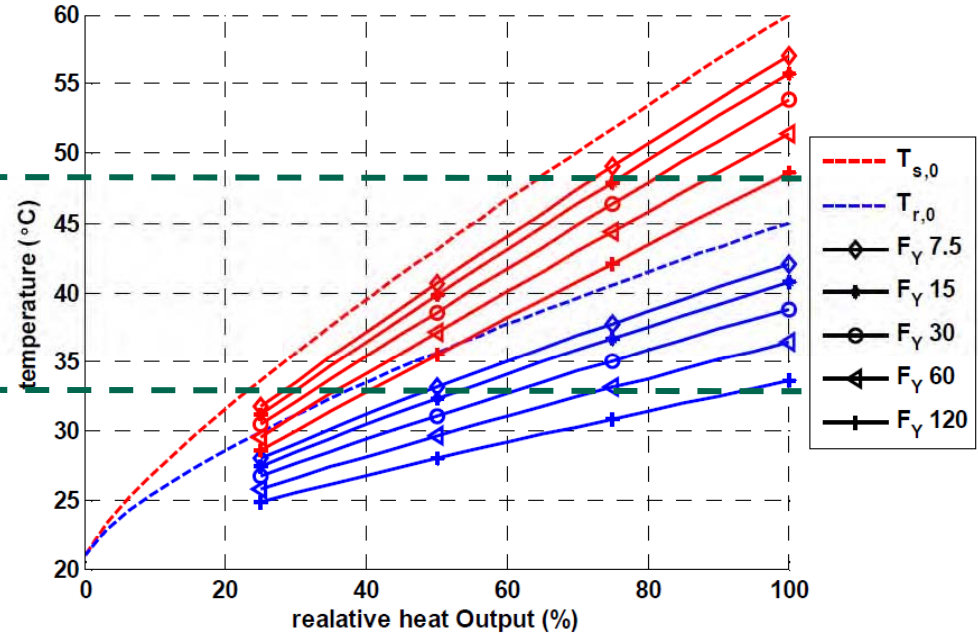
Panel radiator $P_{fan} = 2.7 \text{ W}$



$$\Delta p \approx 2 \text{ Pa} \rightarrow V = 0,074 \text{ m}^3/\text{s}$$

Simulation

Control strategy: m_s unchanged



$$F_Y=60 \rightarrow V = 0,073 \text{ m}^3/\text{s} \quad P = 0,081 \text{ W} \quad P_{fan}=0.8 \text{ W}$$

$$F_Y=120 \rightarrow V = 0,104 \text{ m}^3/\text{s} \quad P = 0,231 \text{ W} \quad P_{fan}=2.3 \text{ W}$$

Thank you for your attention

This work is part of the Primary Energy Efficiency project of Nordic Energy Research.

Per-Olof Johansson
Dept. for Energy sciences
Lund University

Per-Olof.johansson@energy.lth.se

