



Fluid-structure interaction modeling of Air bearing H.R. Javani<sup>1</sup>, P. Kagan<sup>2</sup>, F. Huizinga<sup>1</sup>
1. ASML, MAOM, De Run, Veldhoven, The Netherlands
2. ASML, MTD, De Run, Veldhoven, The Netherlands



### Balance of load with pressure



 $P_{_{Tank}}$ 

### Tilt stiffness



An eccentric load will cause moment which the Air mount should have necessary rotational stiffness to avoid contact and consequently friction.







**ASML** 

Air thin film



1. Flow in the thin film: "Thin-Film Flow, Shell(tffs)"

$$\frac{\partial(\varGamma h)}{\partial t} + \nabla .(h\varGamma v) - \varGamma(v_w \cdot \nabla h_w + v_b \cdot \nabla h_b) = 0$$

1. Flow in the Nozzle: "Edge-ODEs"



2. Structural deformation: "Solid Mechanics"



# Solving physics 1 & 2





## Solving physics 1 & 2



Air consumption = 1.647 nl/min

#### **ASML**

#### Air bearing FSI in COMSOL (Nozzles)



### Results (thin film pressure and flow consumption)



# Results (Animated deformation)



### Results (minimum gap as a function of moment)



#### Conclusions

- COMSOL is relatively more flexible than Ansys in modeling Air bearing.
- The computational time is significantly reduced due to the coupled approach.
- Air bearing model is a highly compressible flows which interact with very stiff structures. These types of problems are difficult to solve using the Iterative FSI Coupling.